

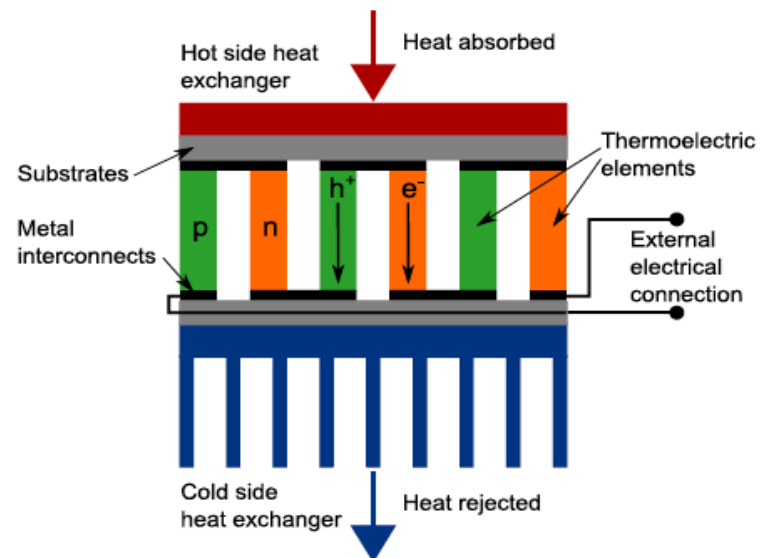
# Thermoelectric Device Engineering

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# Thermoelectric Applications

## Solid State Advantage

- No moving parts
- No maintenance
- Long life
- Scalability



## Cooling - Thermal Management

- Small Refrigerators
- Optoelectronics
- Detectors



## Power Generation (heat to electricity)

### Spacecrafts

Voyager nearly 40 years!

Remote power sources

## Future Possibilities

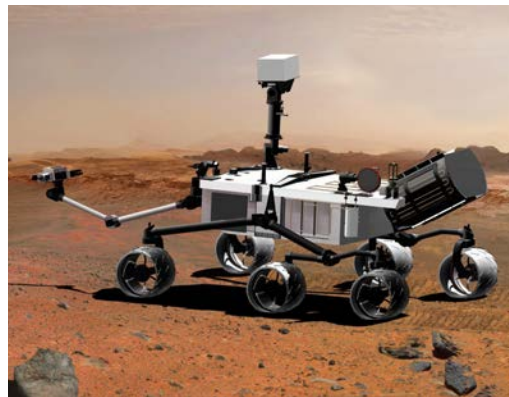
Waste Heat Recovery

Automobiles

Distributed Thermal Management



Saturn Orbiter Cassini

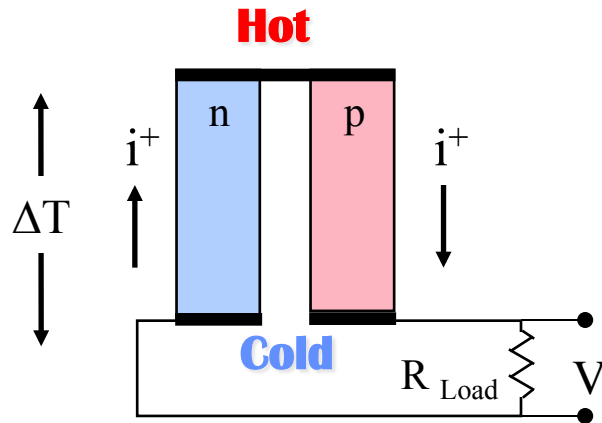


2012 Mars Rover Curiosity



# Power Generation

Draw power through a load



Voltage :  $V = \alpha \Delta T$

Power :  $IV = V^2/R = \alpha^2 \sigma \times \Delta T^2 \times \frac{A}{l}$

Materials Parameters

Power Factor:  $\alpha^2 \sigma$

Electrical conductivity  $\sigma = 1/\rho$

Maximum Efficiency

$$\frac{\text{Electrical Power}}{\text{Heat Removed}} \approx \frac{\alpha^2 \sigma \cdot \Delta T^2}{\kappa \cdot \Delta T + \alpha T_h I + \frac{I^2 \rho}{2}}$$

$\kappa$  = Thermal conductivity

Thermal short, reduces efficiency

Complete Result (approximate)

$$\text{Efficiency} \approx \frac{\Delta T}{T_h} f(zT)$$

Carnot efficiency

$$f \approx \frac{\sqrt{1+zT} - 1}{\sqrt{1+zT} + \frac{T_c}{T_h}}$$

Materials figure of merit

$$zT = \frac{\alpha^2 \sigma}{\kappa} T = \frac{\alpha^2}{\rho \kappa} T$$

# $zT$ vs. $ZT$

## Materials figure of merit $zT$

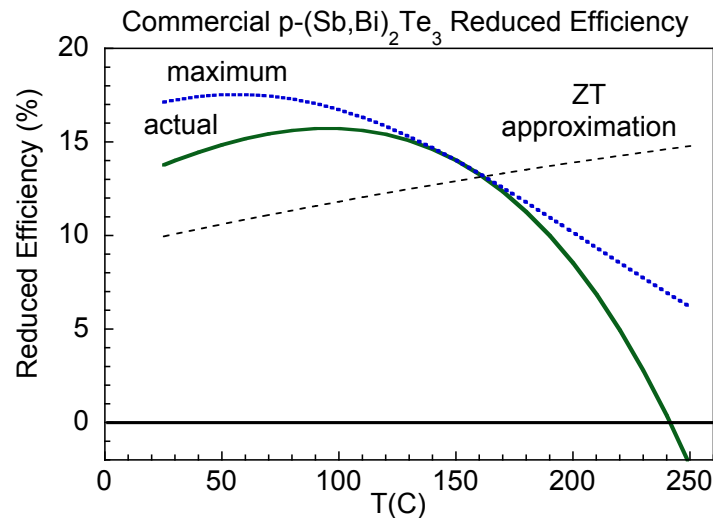
$$zT = \frac{\alpha^2}{\rho\kappa} T$$

Determines maximum reduced efficiency at any given point

$$\max \eta_r = \frac{\sqrt{1 + zT} - 1}{\sqrt{1 + zT} + 1}$$

Actual efficiency is less

- series current not optimal everywhere

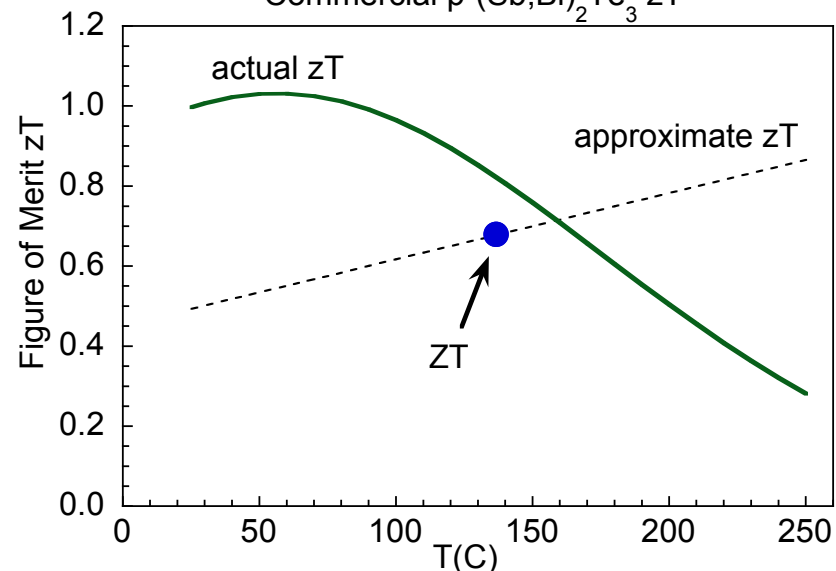


## Device Figure of Merit $ZT$

$$\eta = \frac{\Delta T}{T_h} \cdot \frac{\sqrt{1 + ZT} - 1}{\sqrt{1 + ZT} + T_c/T_h}$$

$ZT = zT$  is approximation for  $\alpha(T)$ ,  $\rho(T)$ ,  $\kappa(T)$ , are constant

Commercial p-(Sb,Bi)<sub>2</sub>Te<sub>3</sub>  $zT$

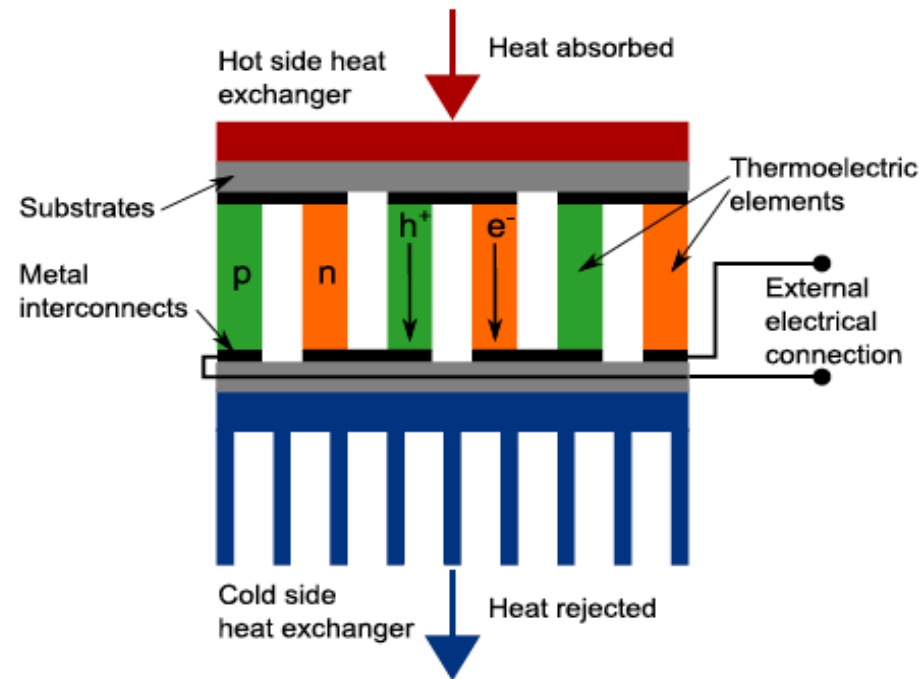


Real materials:  $zT \neq ZT$

especially  $\max zT \neq ZT$

Beware conclusions about changing  $\Delta T$

# Thermal Impedance Match





# Small $\Delta T$ Thermoelectric Generator

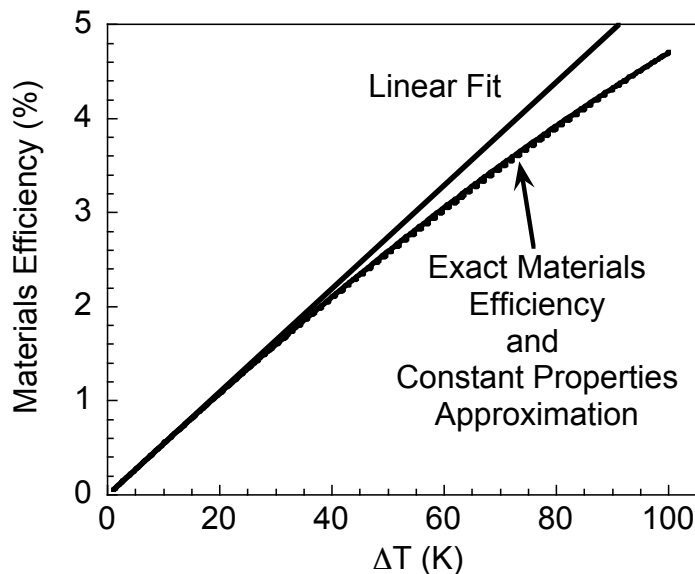
Efficiency proportional to  $\Delta T$

$$\eta = \eta_1 \cdot \Delta T$$

$$\eta_1 \approx 0.05\% / K$$

Power requires Heat flow

$$P = \eta Q$$



Heat Flow related to thermal conductance of TE

$$Q_h = K\Delta T + SIT_h - \frac{1}{2}I^2R$$

$$Q \approx K_{eff}\Delta T$$

$$K = \kappa_{eff} \frac{A}{l}$$

$P$  - electrical power out

$$K_{eff} \approx K \cdot \sqrt{1 + Z\bar{T}}$$

$Q$  - Heat input

$\eta$  - Efficiency

$K$  - Thermal Conductance of TE

$\kappa$  - Thermal Conductivity

$A$  - Area of TE

$l$  - length of TE

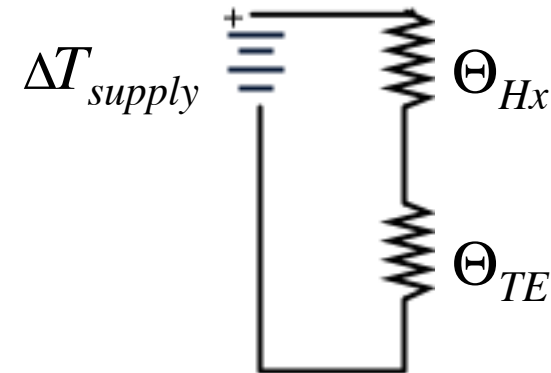
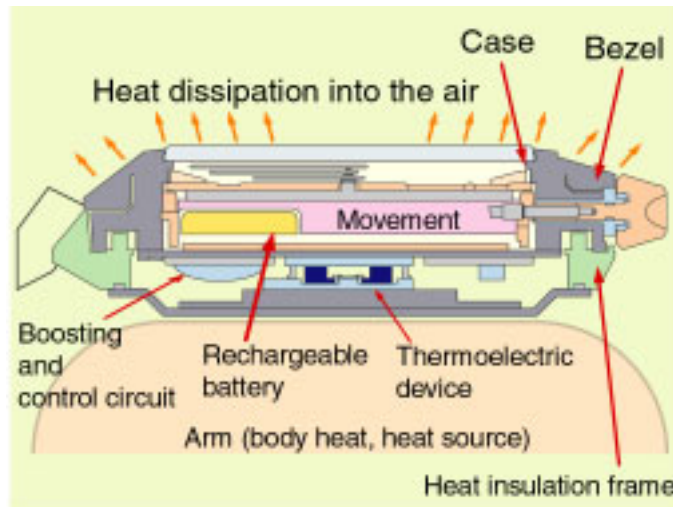
$$\eta = \frac{\Delta T}{T_h} \cdot \frac{\sqrt{1 + Z\bar{T}} - 1}{\sqrt{1 + Z\bar{T}} + T_c/T_h}$$



# Thermal Impedance Matching

Claim: 'All the heat you need'

Reality: Space limits thermal impedance of source + sink



$$Q = \frac{\Delta T_{\text{supply}}}{\Theta_{Hx} + \Theta_{TE}} = \frac{\Delta T_{TE}}{\Theta_{TE}}$$

$$P = \eta_1 \Delta T_{\text{supply}}^2 \frac{\Theta_{TE}}{(\Theta_{Hx} + \Theta_{TE})^2}$$

Example: air cooled TEG

Typical air Heat Exchanger

- $\sim 0.2 \text{ W/K cm}^2$

Maximum power when thermal impedance is matched

- $\Theta_{TE} = \Theta_{Hx}$

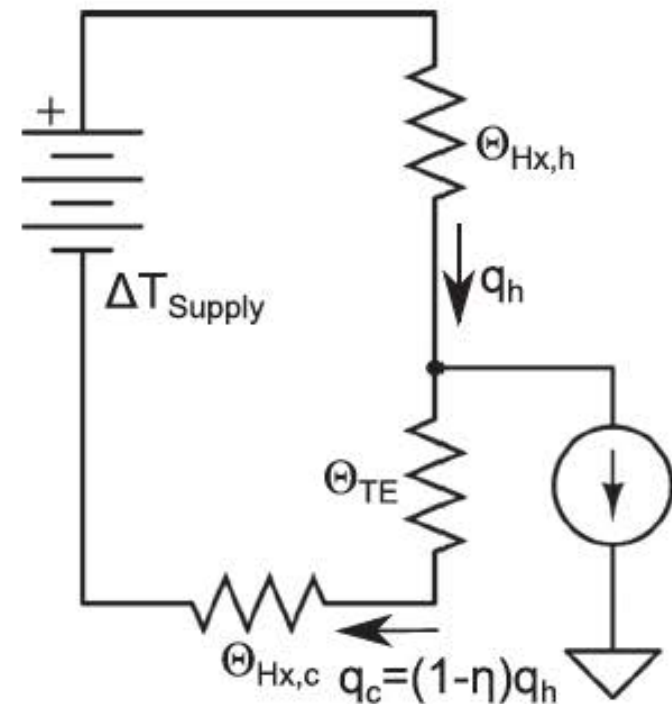
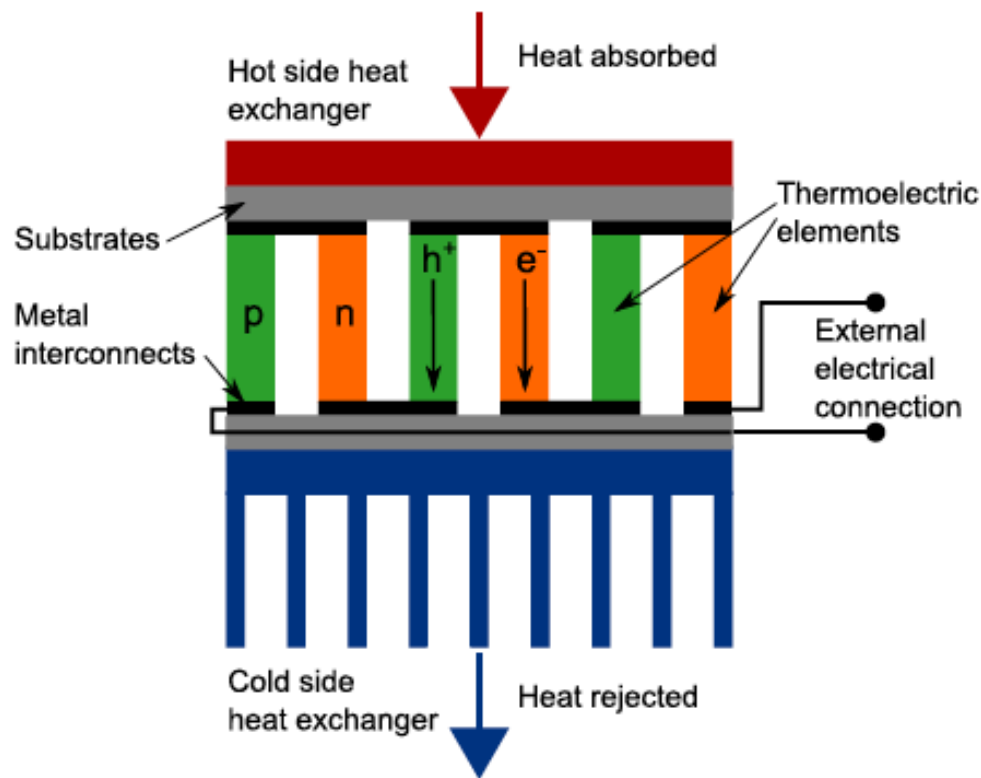
$$\frac{\partial P}{\partial \Theta_{TE}} = 0 \Rightarrow \Theta_{TE} = \Theta_{Hx}$$



# Effective Thermal Conductivity

Thermal model

Thermal Circuit

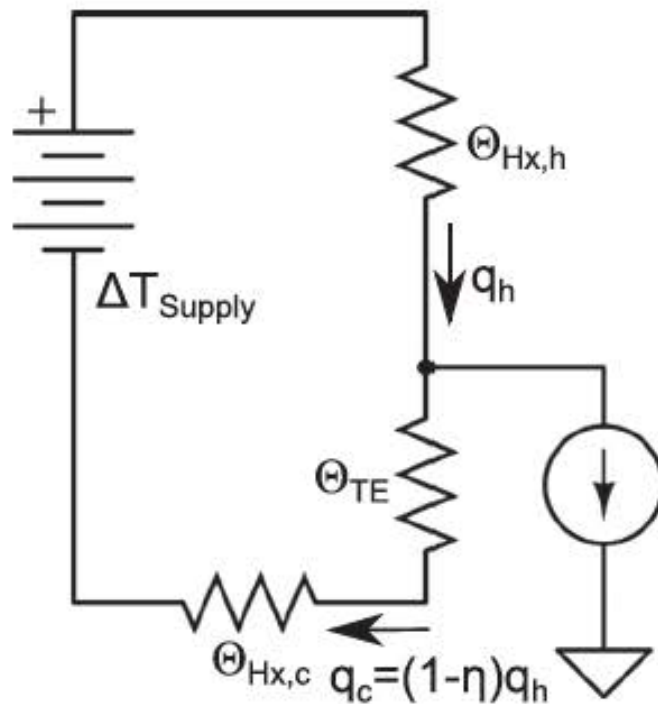




# Thermal Circuit Analysis

Thermal Circuit

Mathematical Model



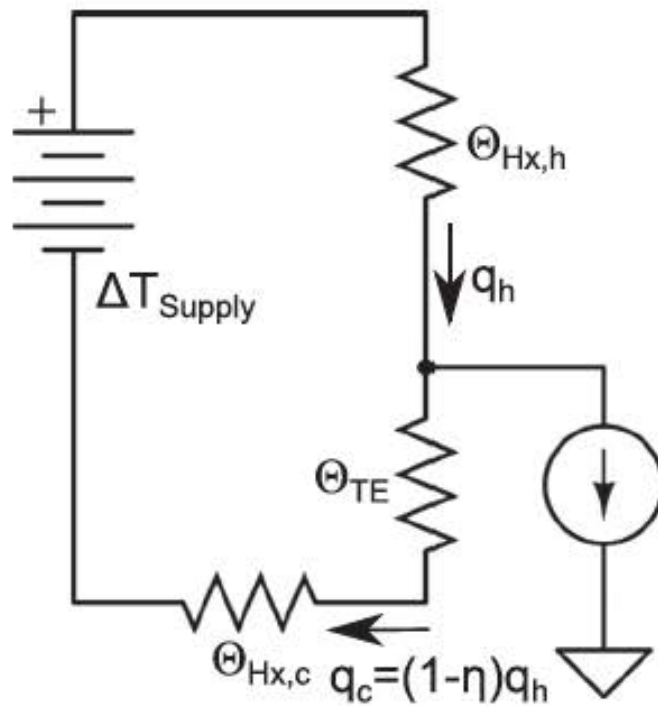
$$\Delta T_{Hx} = q_h(\Theta_{Hx,h} + (1 - \eta)\Theta_{Hx,c}) = q_h\Theta_{Hx}$$

$$\Delta T_{TE} = \Delta T_{\text{supply}} \frac{\Theta_{TE}}{\Theta_{Hx} + \Theta_{TE}}$$

# Power and Efficiency

## Thermal Circuit

## Thermoelectric Definitions



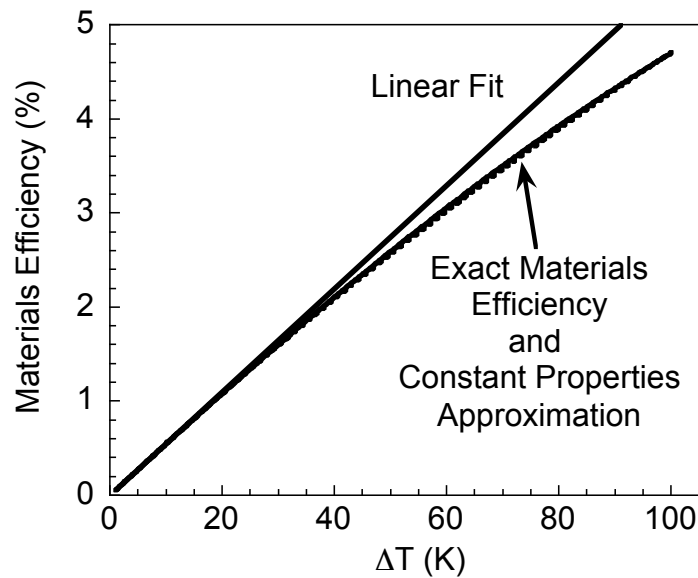
$$P = \eta q_h$$

$$\eta = \frac{\Delta T_{\text{TE}}}{T_h} \eta_{r,d}$$

$$\eta = \frac{\Delta T_{\text{TE}}}{T_h} \frac{\sqrt{1 + ZT} - 1}{\sqrt{1 + ZT} + T_c/T_h}$$

# Efficiency dependence on $\Delta T$

## Mathematical Model



$$\eta_{r,lmax} = \frac{\sqrt{1 + z\bar{T}} - 1}{\sqrt{1 + z\bar{T}} + 1}$$

$$\eta = \frac{\Delta T_{TE}}{T_h} \frac{\sqrt{1 + Z\bar{T}} - 1}{\sqrt{1 + Z\bar{T}} + T_c/T_h}$$

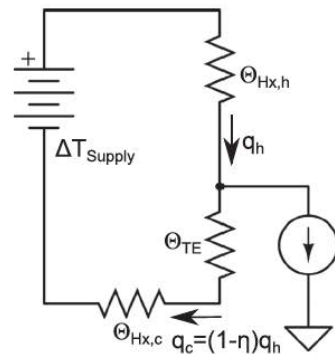
$$\frac{P_{max}}{A_{Hx}} = \frac{\Delta T_{supply}^2 h_{Hx}}{4T_h} \frac{\sqrt{1 + Z\bar{T}} - 1}{\sqrt{1 + Z\bar{T}} + T_c/T_h}$$

$$\eta_{r,d} = \frac{2T_h \left(1 - \left(\frac{T_c}{T_h}\right)^{\eta_{r,lmax}}\right)}{\Delta T_{supply}}$$

$$\frac{P_{max}}{A_{Hx}} = \frac{\Delta T_{supply} h_{Hx}}{2} \left(1 - \left(\frac{T_c}{T_h}\right)^{\eta_{r,lmax}}\right)$$

# Thermal Impedance Match

## Thermal Circuit



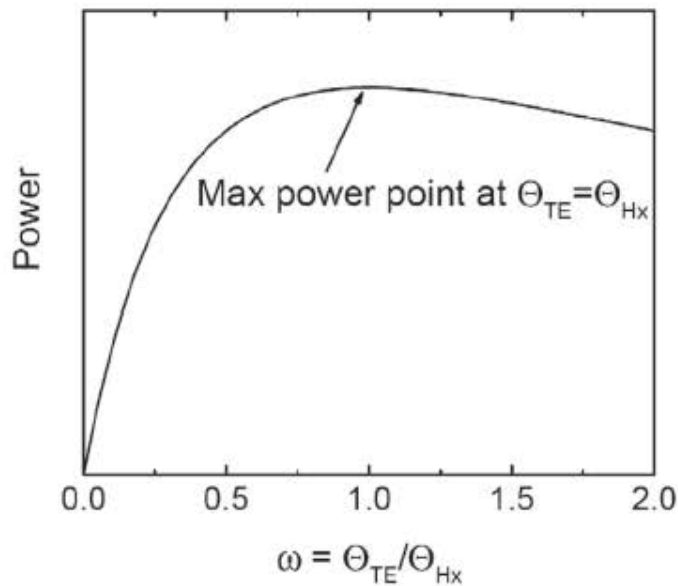
## Thermal Impedance match derivation

$$\Delta T_{TE} = \Delta T_{supply} \frac{\Theta_{TE}}{\Theta_{Hx} + \Theta_{TE}}$$

$$\Delta T_{supply} = q_h(\Theta_{Hx} + \Theta_{TE})$$

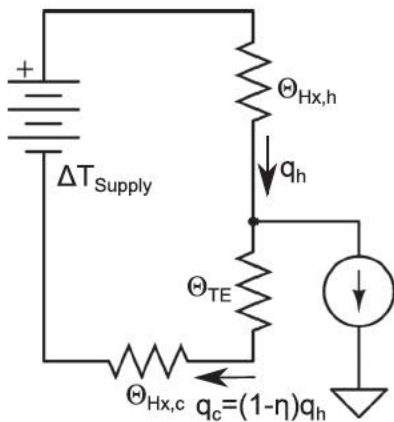
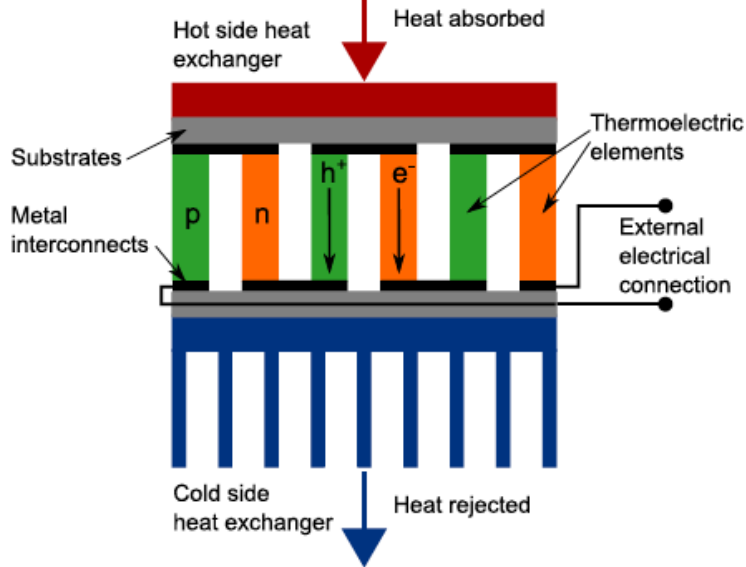
$$P = \frac{\Delta T_{supply}^2 \eta_{r,d}}{T_h} \frac{\Theta_{TE}}{(\Theta_{Hx} + \Theta_{TE})^2}$$

$$P_{max} = \frac{\Delta T_{supply}^2 \eta_{r,d}}{4T_h \Theta_{Hx}}$$



# Effective Thermal Conductivity

## Thermal model



## Area Specific Formulation

Cross sectional Area  $A$

$$h_{Hx} = 1/\Theta_{Hx}A_{Hx}$$

forced air

forced water

$$h_{Hx} \approx 0.004 \text{ W/cm}^2\text{K}$$

$$h_{Hx} \approx 0.6 \text{ W/cm}^2\text{K}$$

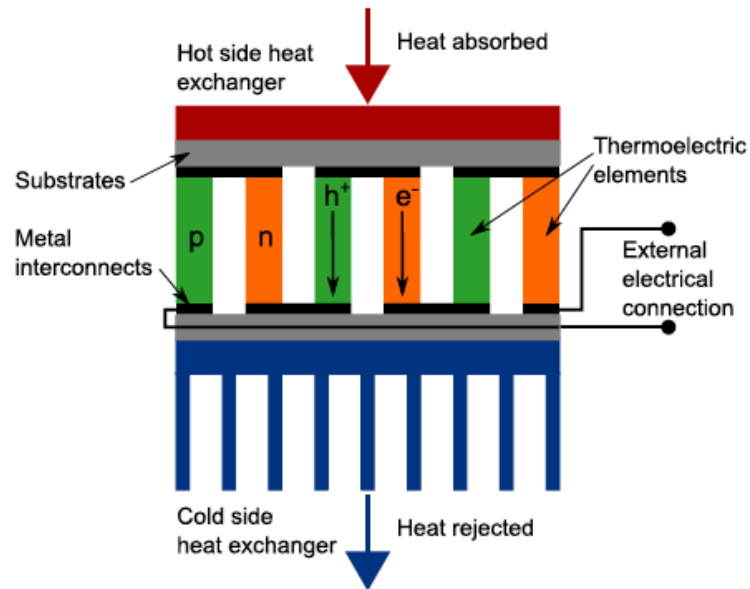
$$\frac{P_{max}}{A_{Hx}} = \frac{\Delta T_{supply}^2 h_{Hx} \eta_{r,d}}{4T_h}$$

# Effective Thermal Conductivity

Thermal model

Area Specific Formulation

Cross sectional Area  $A$



$$\Theta_{TE} = \frac{l}{\kappa_{eff} A_{TE}}$$

filling factor:  $f = A_{TE} / A_{Hx}$

$$\frac{1}{h_{Hx}} = \Theta_{Hx} A_{Hx} = \Theta_{TE} A_{Hx} = \Theta_{TE} \frac{A_{TE}}{f}$$

$$l = \frac{f \kappa_{eff}}{h_{Hx}}$$

# Effective Thermal Conductivity

Effective  $\kappa$  at point of optimum operation

Heat flux  $q''$

$$q''_h = \frac{\kappa_{\text{eff}}}{l} (T_h - T_c)$$

Compatibility formalism

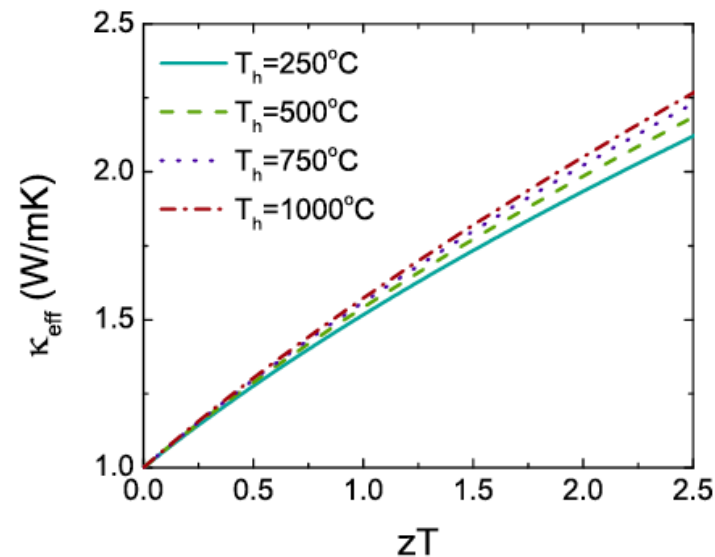
$$q''_h = \frac{\Phi_h \int_{T_c}^{T_h} \kappa u dT}{l}$$

gives

$$\kappa_{\text{eff}} = \frac{\Phi_h}{(T_h - T_c)} \int_{T_c}^{T_h} \kappa u dT$$

$\kappa_{\text{eff}}$  Approximate Forms

$$\kappa_{\text{eff}} \approx \kappa \cdot \sqrt{1 + z\bar{T}}$$



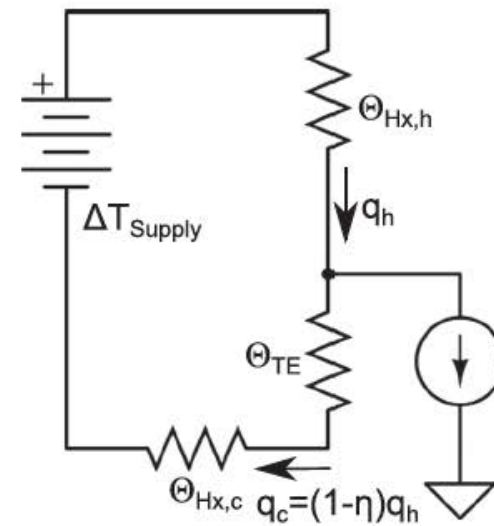
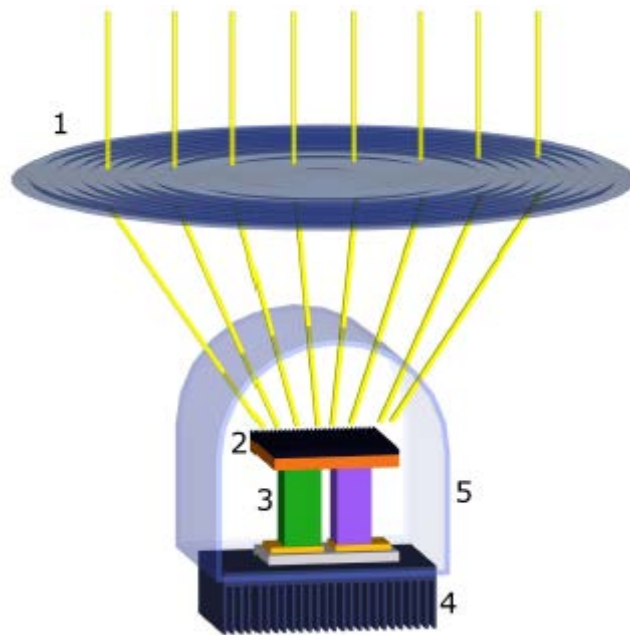
$$\kappa_{\text{eff}} = \frac{\kappa T_h (1 + z\bar{T} + \sqrt{1 + z\bar{T}})}{2(T_h - T_c)} \left( 1 - \left( \frac{T_h}{T_c} \right)^{k_g} \right)$$

$$k_g = \frac{2 - 2\sqrt{1 + z\bar{T}}}{z\bar{T}}$$



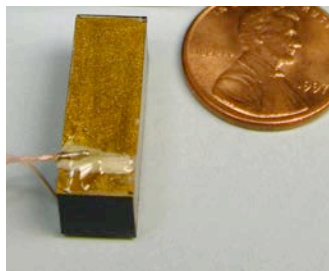
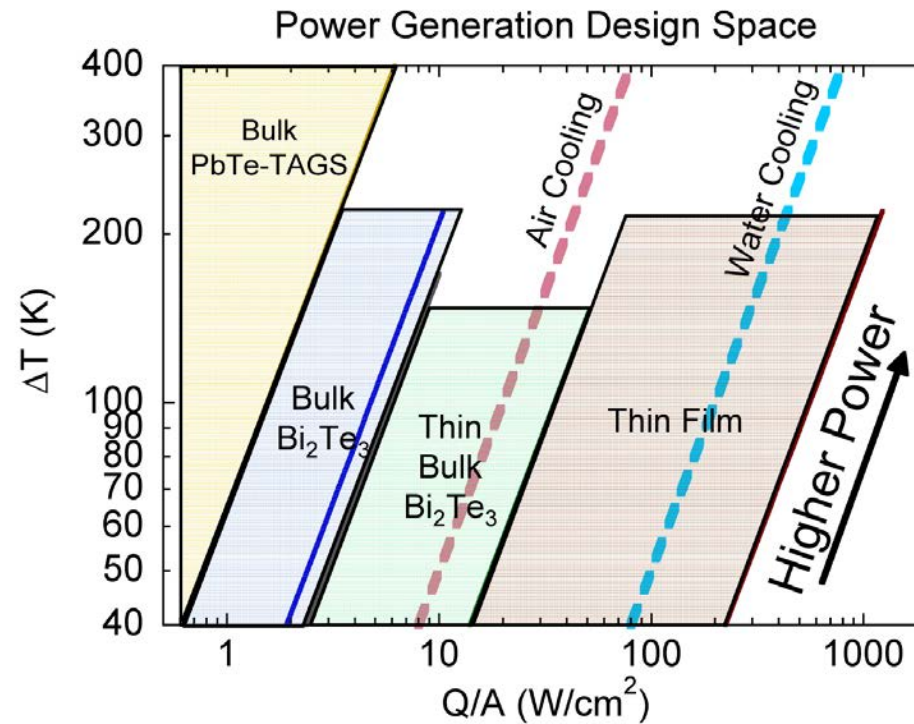
# Solar Thermoelectric Example

## Thermal model



$$\eta_{STEG} = \frac{(T_h - T_c) \kappa_{eff}}{q_{inc}'' L_{th}} \left( 1 - \left( \frac{T_c}{T_h} \right)^{\eta_r} \right)$$

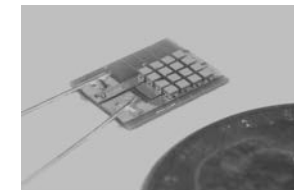
# TE phase space



Thermopile  
 $l \sim 20$  mm



Commercial Module  
 $l \sim 2$  mm



Thin Film Device  
 $l \sim 0.02$  mm

# Example - Wristwatch

How much power could you harvest from body Heat?

Body ( $2\text{m}^2$ ) releases  $100\text{W}$  heat

- $20\text{ mW}/\text{cm}^2$

Skin temperature  $33\text{C}$ ,  $RT = 21\text{C}$

- $\Delta T = 12\text{K}$

Thermal Impedance match TE

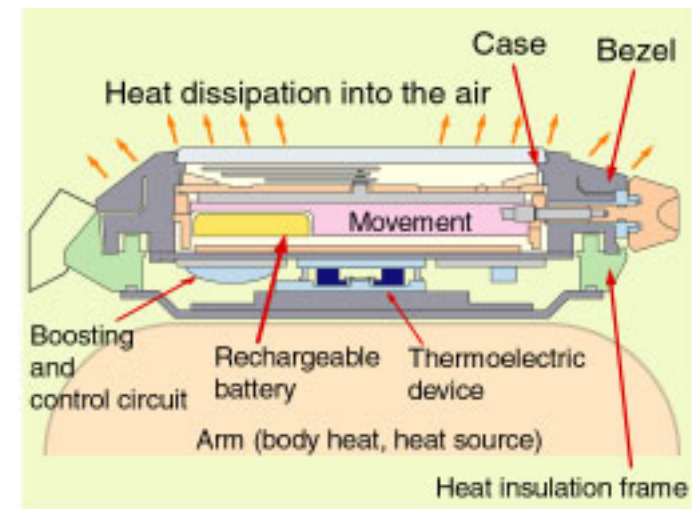
- $\Delta T_{TE} = 6\text{K}$
- Carnot Efficiency = 2%
- Reduced Efficiency ( $ZT \sim 1$ ) = 20%
- Total Efficiency = 0.4%
- $80\text{ uW}/\text{cm}^2$

Seiko Wristwatch

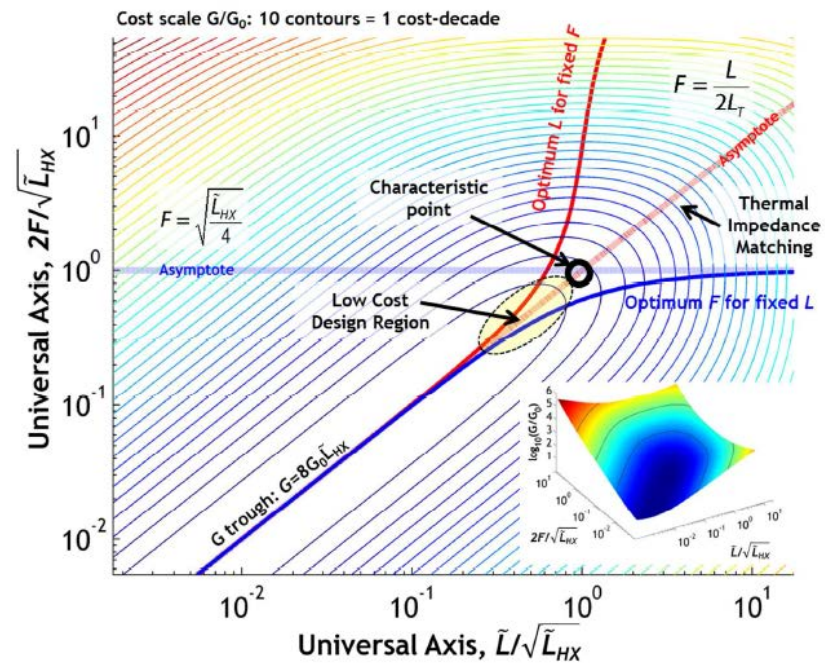
- $22\text{ uW}$  output
- $20\text{mV}/\text{K} * 10$  modules
- $300\text{mV OC} = 1.5\text{K}$  across TE
- $600\text{K}/\text{W} / 10$  modules  
=  $25\text{mW}$  heat  
= 0.1% efficiency



Seiko Thermic, Thermoelectric Powered Wristwatch



# Power and Cost





# TEG Design Optimization

1. Decide type and size of heat exchangers = determine  $h_{Hx}$

roughly 
$$\frac{P_{max}}{A_{Hx}} = \frac{\Delta T_{supply}^2 h_{Hx} \eta_{r,d}}{4T_h}$$
 with  $\eta_{r,d} \sim 0.15$

2. Decide target  $T_h$  and  $T_c$ , roughly  $\Delta T_{TE} = \Delta T_{supply}/2$

Determine TE materials used

3. Determine TEG target length  $l$  and fill factor  $f$  from

$$l = \frac{f \kappa_{eff}}{h_{Hx}}$$

Determine TE materials used

4. Optimization of module based on optimum efficiency

Number of couples from Target Voltage (Current from V and P)

Area of elements from optimum area ratio

Load Resistance approx

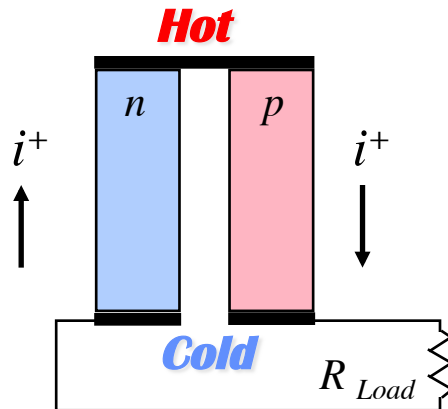
$$R_{Load} = R\sqrt{1 + ZT}$$

$$\frac{A_p}{A_n} \approx \sqrt{\frac{\rho_p \kappa_n}{\rho_n \kappa_p}}$$



# Power from TEG

## Power derivation



$$P = I^2 R_L$$

$$I = \frac{\alpha \Delta T}{R_{TE} + R_L}$$

$$P = \frac{\alpha^2 \Delta T^2}{(R_L + R_{TE})^2} R_L$$

$$m = R_L / R_{TE}$$

$$P = \frac{\alpha^2 \Delta T^2 A}{\rho l} \frac{m}{(1 + m)^2}$$

# Power from TEG

$$P = \frac{\alpha^2 \Delta T^2 A}{\rho l} \frac{m}{(1+m)^2}$$

Maximum power ?

mathematically this function has no maximum!

A = infinity

- consider power density  $P/A$

$\Delta T$  = maximum?

- actually one should thermal impedance match  $\Delta T \sim \Delta T_{supply} / 2$

$l$  = zero

- set  $l$  to a constant ?!?

–off the shelf component? = how to get most power from given TEG  
are we optimizing the use of a given TEG or design a TEG for application?

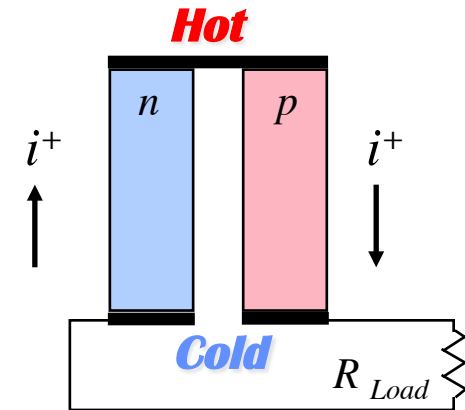
$$\frac{\partial}{\partial m} \left( \frac{P}{A} \right)_{l, \alpha, \rho, \Delta T} = 0 \Rightarrow m = 1$$

used to derive electrical load match condition  $m = 1$

*Not mathematically rigorous* when  $l$  is design parameter !

used to derive power factor  $\alpha^2/\rho$  more important than  $zT$  for power generating

**NO! Faulty derivation – this function has no maximum! beware of limits to infinity**





# TEG Power Competition

Max power TE

$T_h$  and  $T_c$ , with  $q_h$  heat through it

$$R_{Load} = R$$

has higher current

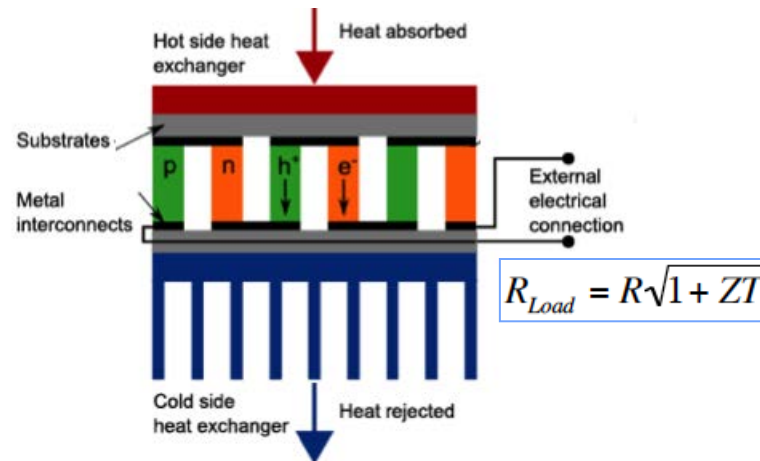
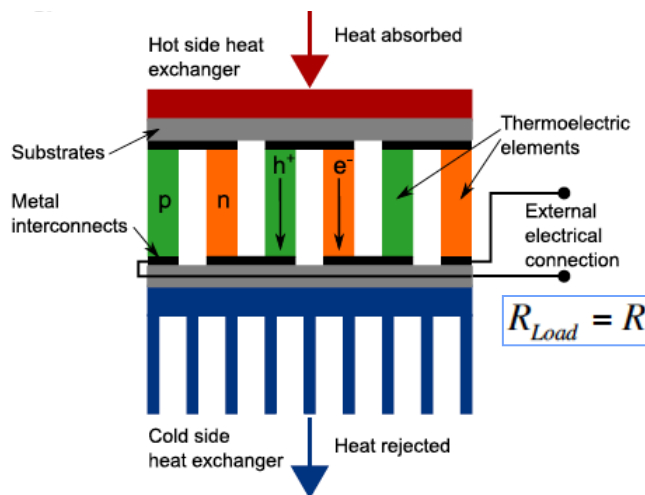
Max efficiency TE with same

$T_h$  and  $T_c$ , with  $q_h$  heat through it

$$R_{Load} = R\sqrt{1 + ZT}$$

has lower current so reduce  $l$  to keep  $q_h$  as Max power TE

$$q_h'' = \frac{K_{eff}}{l} (T_h - T_c)$$



# TEG Power Competition

Max power TE

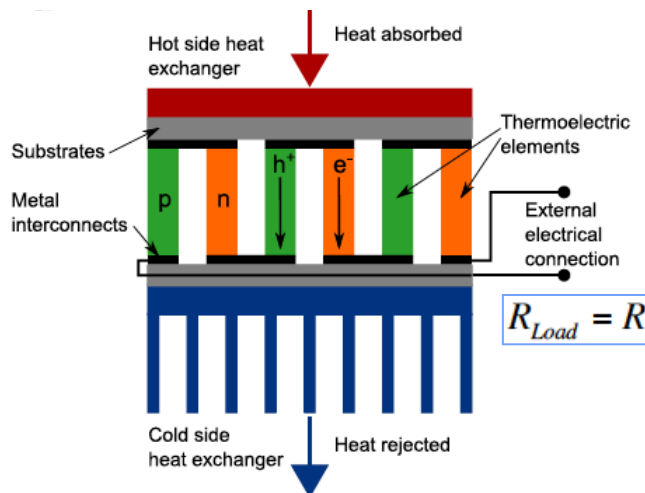
$T_h$  and  $T_c$ , with  $q_h$  heat through it

$$R_{Load} = R$$

has lower efficiency

$$P = \eta q_h$$

therefore **lower** power



Max efficiency TE with same

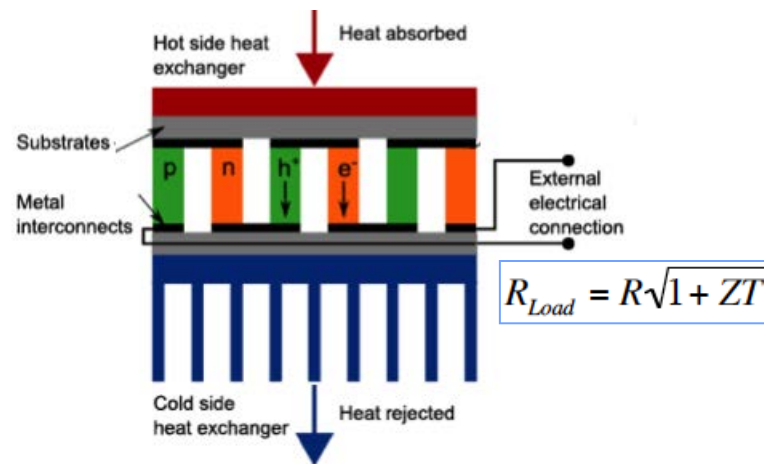
$T_h$  and  $T_c$ , with  $q_h$  heat through it

$$R_{Load} = R\sqrt{1 + ZT}$$

has higher efficiency

$$P = \eta q_h$$

therefore **higher** power





# TEG Power Competition

Proof by contradiction

where TE height  $l$  is design variable

1) Suppose:  $R_{Load} = R$  always provides more power than

$$R_{Load} = R\sqrt{1 + ZT}$$

2) give example of  $R_{Load} = R$

3) show counter example where  
provides more power

$$R_{Load} = R\sqrt{1 + ZT}$$

4) supposition 1) must be false



# TEG Power Competition

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Can be used to show maximum power design  
where TE height  $l$  is design variable

$$R_{Load} = R\sqrt{1 + ZT}$$

$$\frac{A_p}{A_n} \approx \sqrt{\frac{\rho_p K_n}{\rho_n K_p}}$$

**$zT$  is figure of merit for power**

not power factor  $\alpha^2 \sigma$  !

high  $\kappa$  matters !

# ZT at any cost?

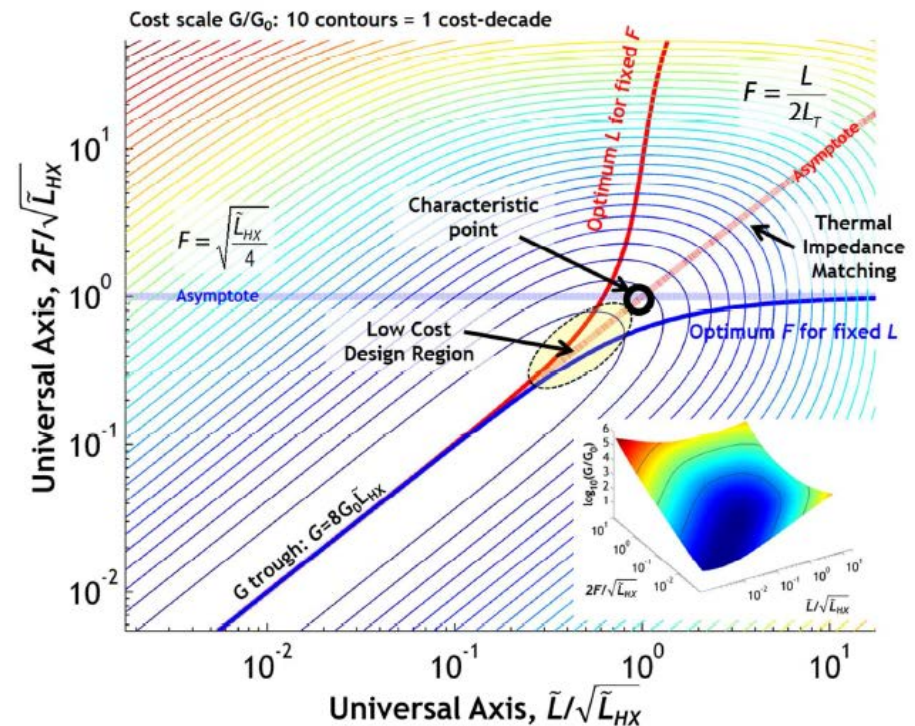
Linear Thermal model using  $\kappa_{eff}$  facilitates cost modeling

lowest system cost/W is usually dominated by heat exchanger cost rather than TE material cost (\$200/kg material)

Power increases with ZT so cost directly depends on ZT but not TE cost

$$\frac{Cost}{W} \approx \left( \frac{Cost}{Area} \right)_{HX} \cdot \frac{1}{ZT}$$

ZT is TE cost metric

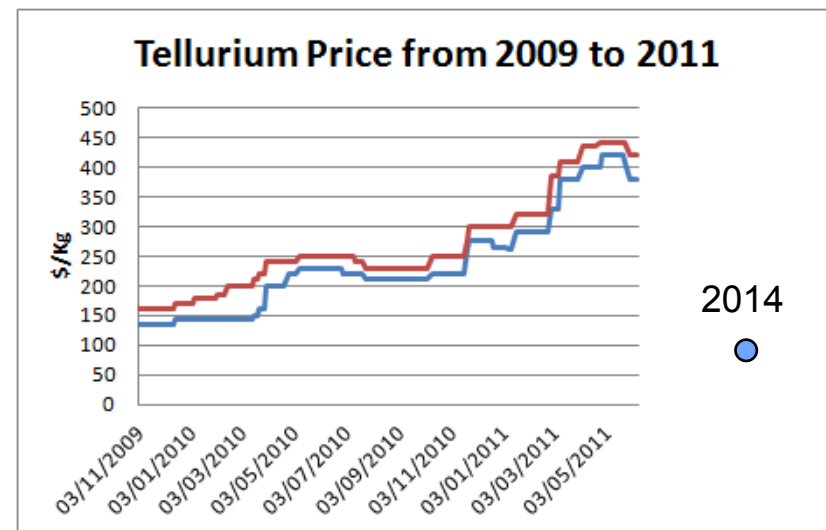
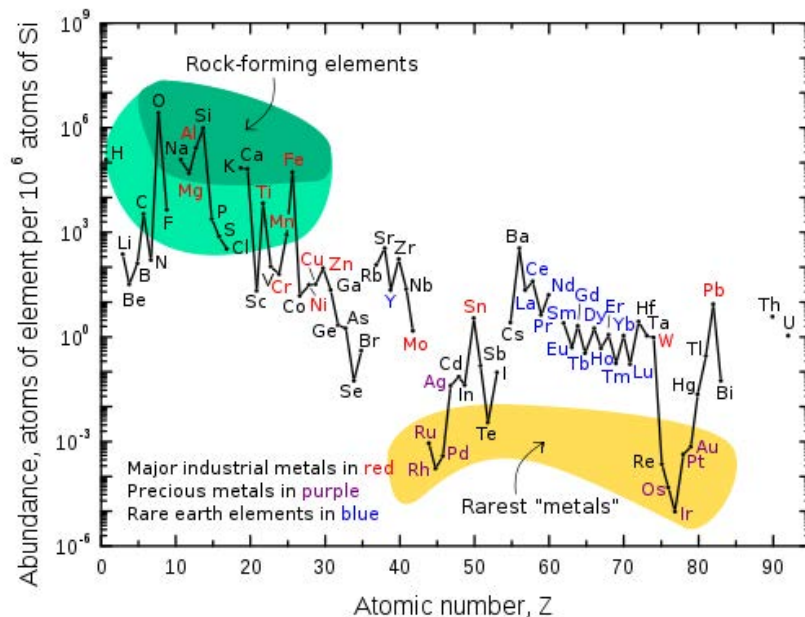


# ZT at any cost?

What about High Cost and abundance of Te ?!?

Te has low abundance on Earth crust

Cost of Te above \$400/kg in 2011 (back to \$100/kg in 2014)



Te is a by product of Cu refining, capacity not at full production

Bi is \$20/kg, Pd is \$25,000/kg