



University
of Glasgow | College of Science
& Engineering

EPSRC – Thermoelectric Network

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Considerations on the Applications of Thermoelectric Devices

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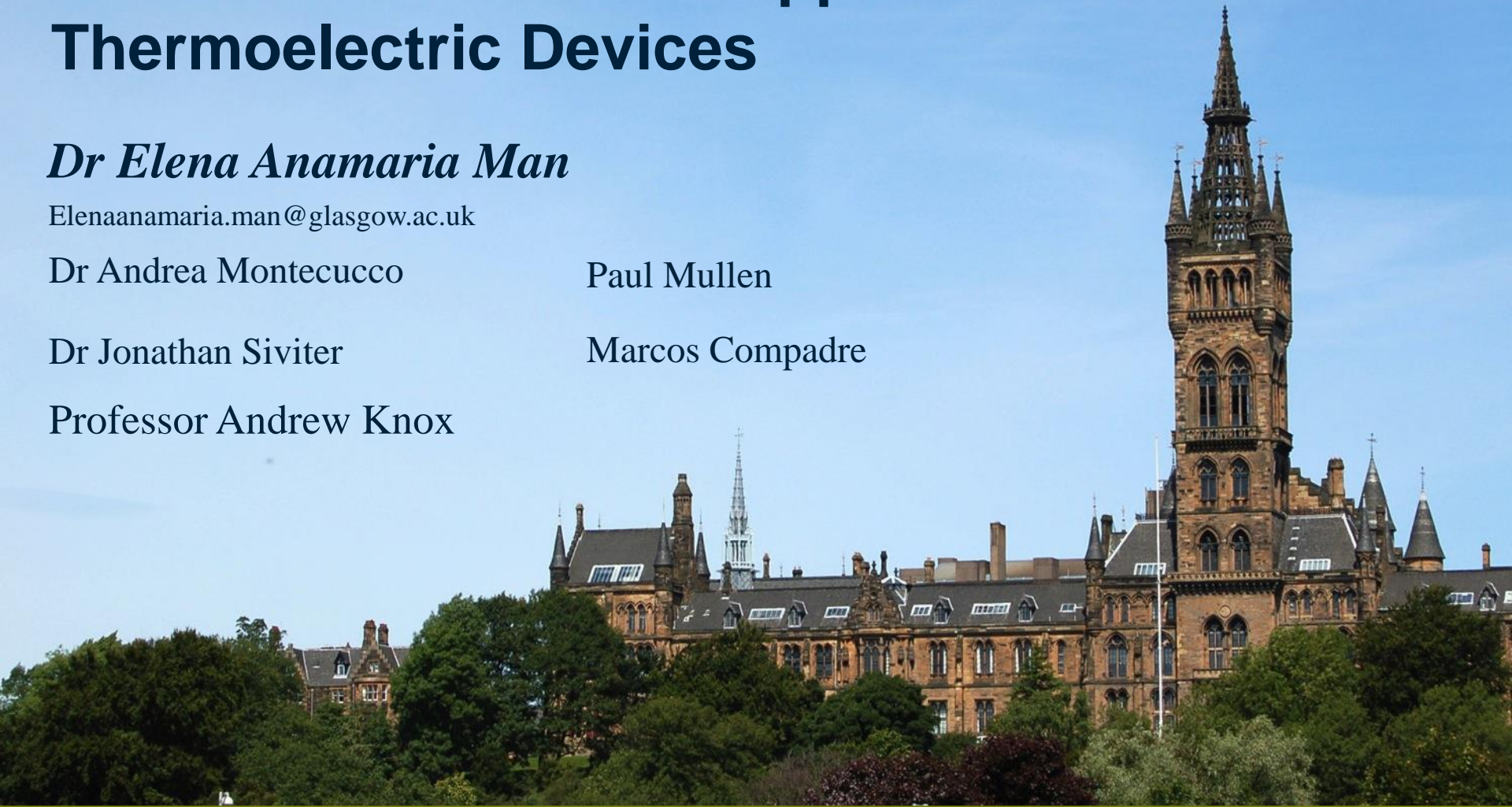
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Professor Andrew Knox





I. Thermoelectric generation

- ❖ Constant temperature vs constant heat TEG characterization.
- ❖ Maximum power point (MPP) in constant heat.
- ❖ TEG test and application Glasgow systems – description and system issues discussion.

II. Thermoelectric cooling

- ❖ THP characterization.
- ❖ Heat battery system – design and system issues.



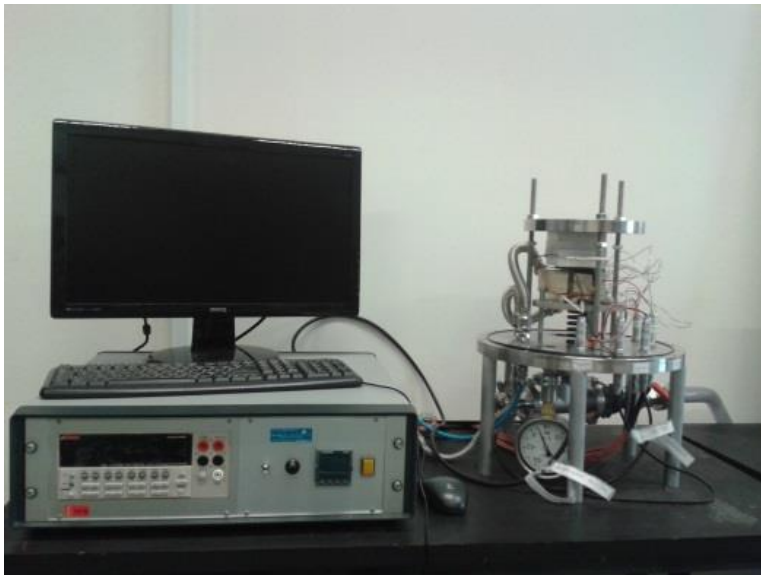
THERMOELECTRIC GENERATION



What does it mean?

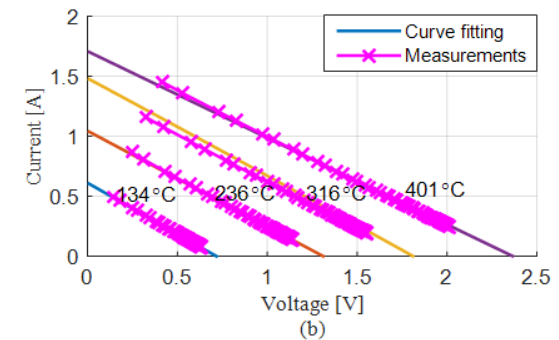
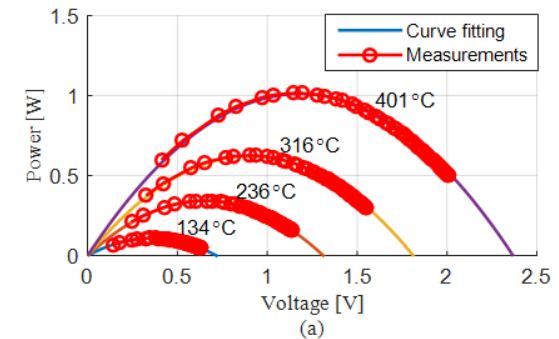
- ❖ Maintaining a constant temperature difference across a TE module using a controllable setup, independent of any other parameters in the system.

How?

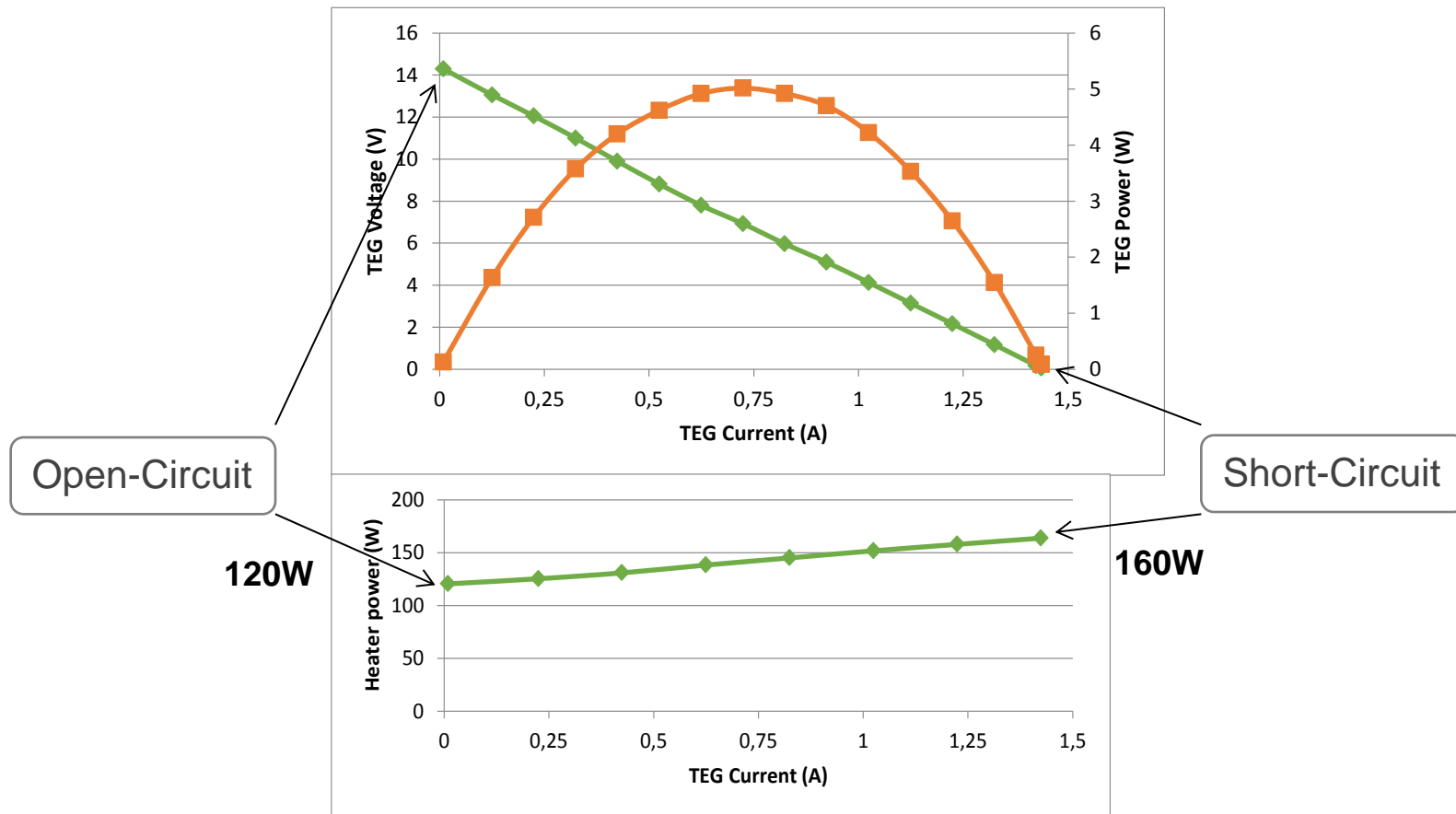


Commercial TE module parameter
characterization setup

Why?



Constant temperature a) P - V and b) I - V
curves of a TE module



Heater power variation during **constant temperature** ($\Delta T = 154 \text{ }^\circ\text{C}$) testing of thermoelectric module from open-circuit to short-circuit condition.

“Constant heat” characterization



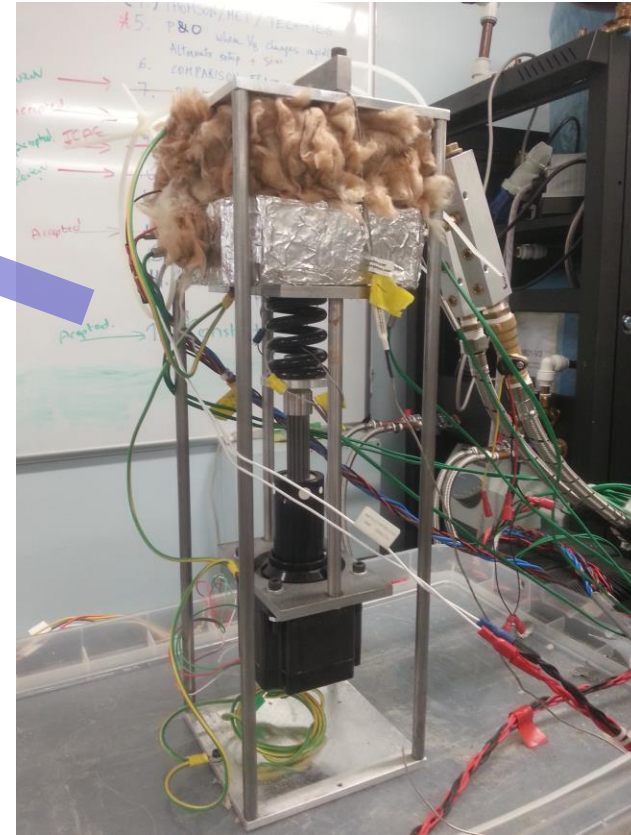
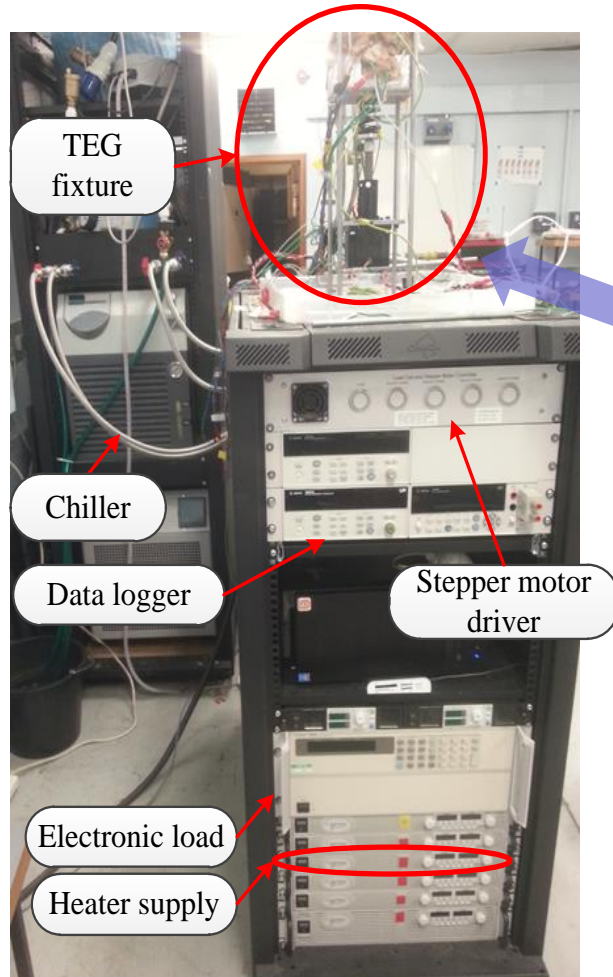
Why ? *Heat available for TEG integration is “limited” – significantly less constant temperature available for load changing conditions.*

How ? *Using a test fixture and controlling the heat flux through the TEG rather than maintaining a constant temperature gradient across the module.*

Target? *Considering the electrical and thermal interaction in the TEG system.*

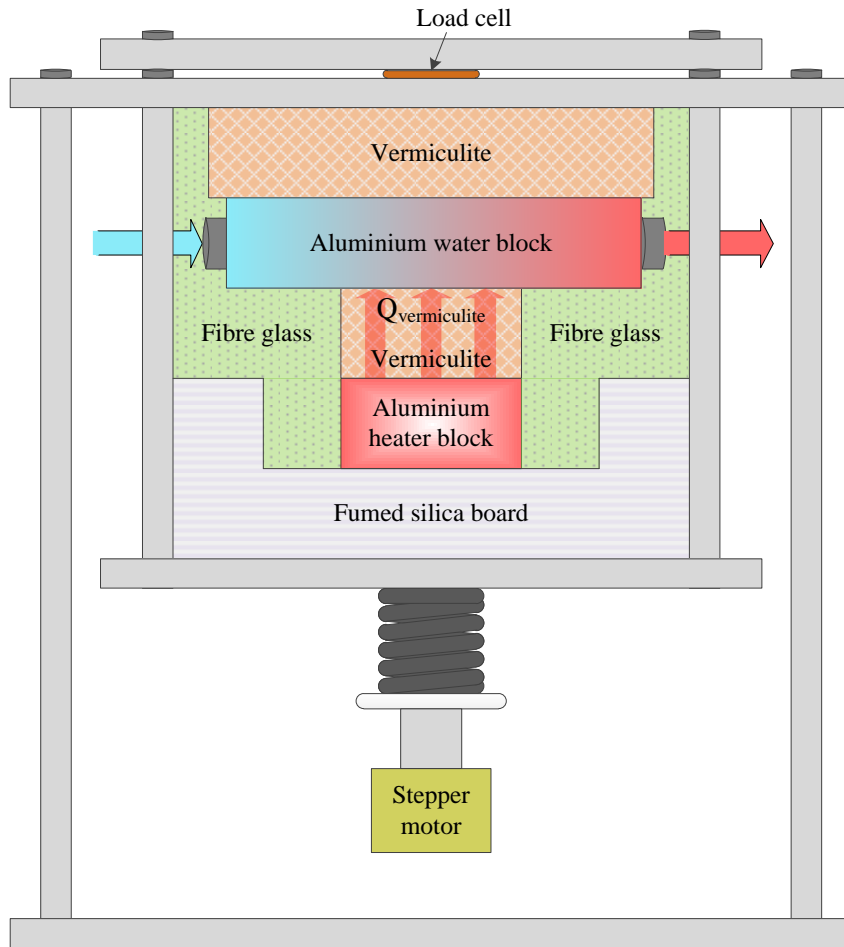


TE module test fixture





Heat loss characterization



$$Q_{loss} = P_{heater} - Q_{vermiculite} = P_{heater} - \kappa_{Verm} A_{Verm} \frac{T_h - T_c}{l_{Verm}}$$

1) P_{heater} – known (set by user).

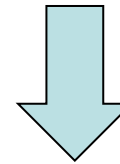
2) $\kappa_{Verm} = 0.07W/m \cdot K$

$A_{Verm} = 0.0016m^2$

$l_{Verm} = 0.025m$

T_h and T_c measured

$Q_{vermiculite}$ can be calculated

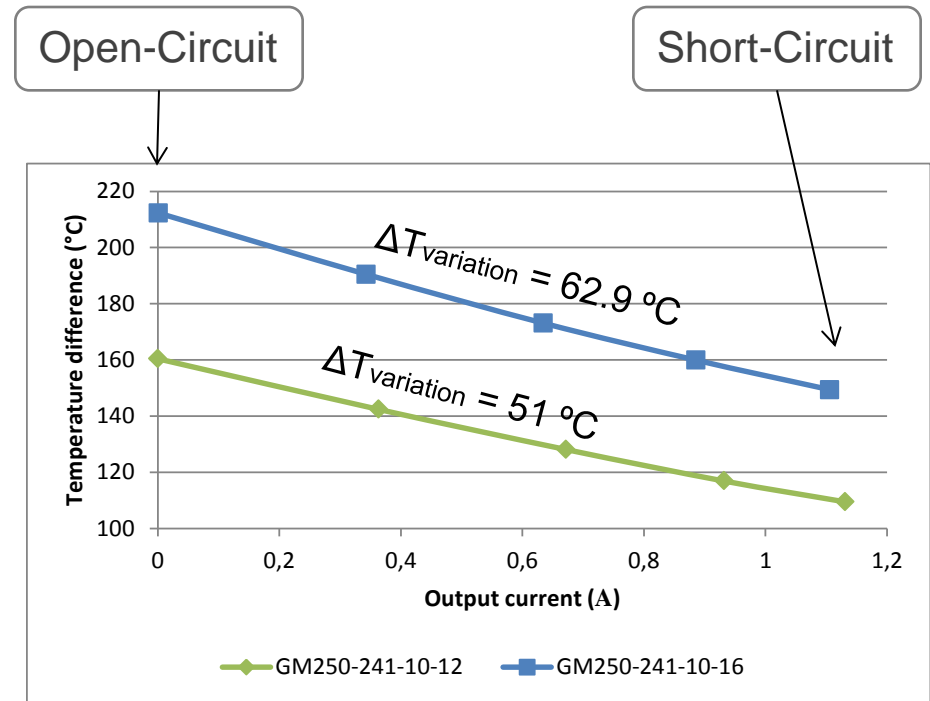
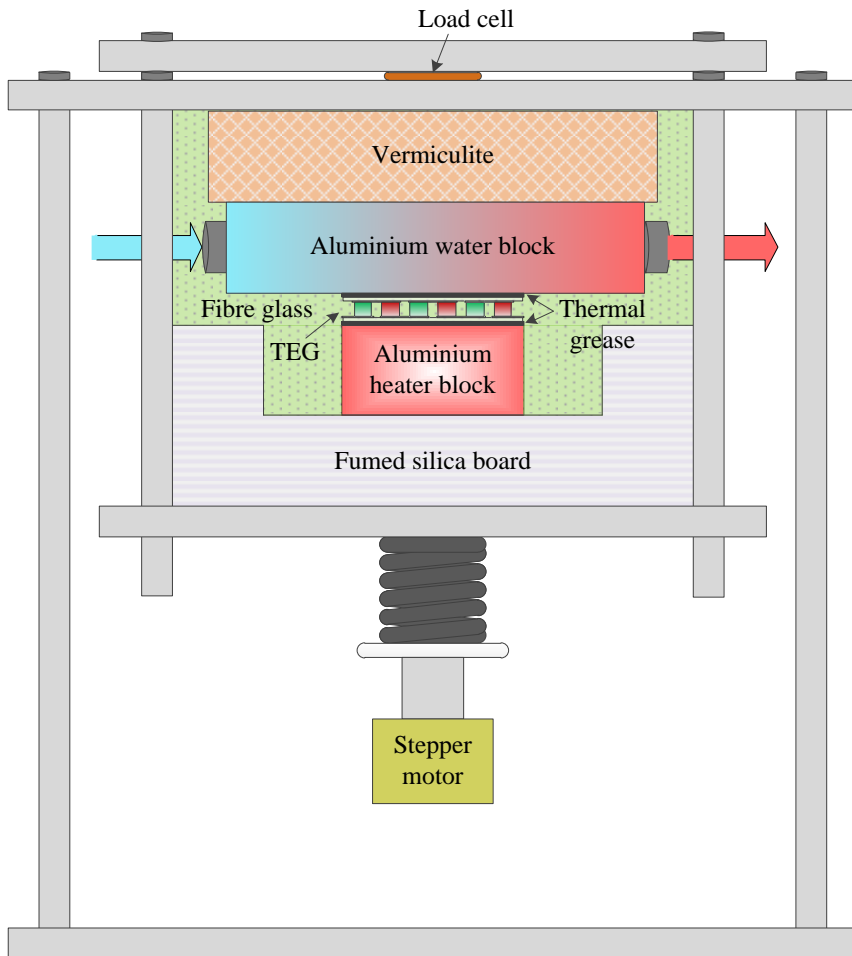


$$Q_{losses_fit} = 3 \cdot 10^{-5} \cdot T_h^2 + 0.0357 \cdot T_h$$

$$Q_{teg} = Q_{heater} - Q_{losses_fit}$$



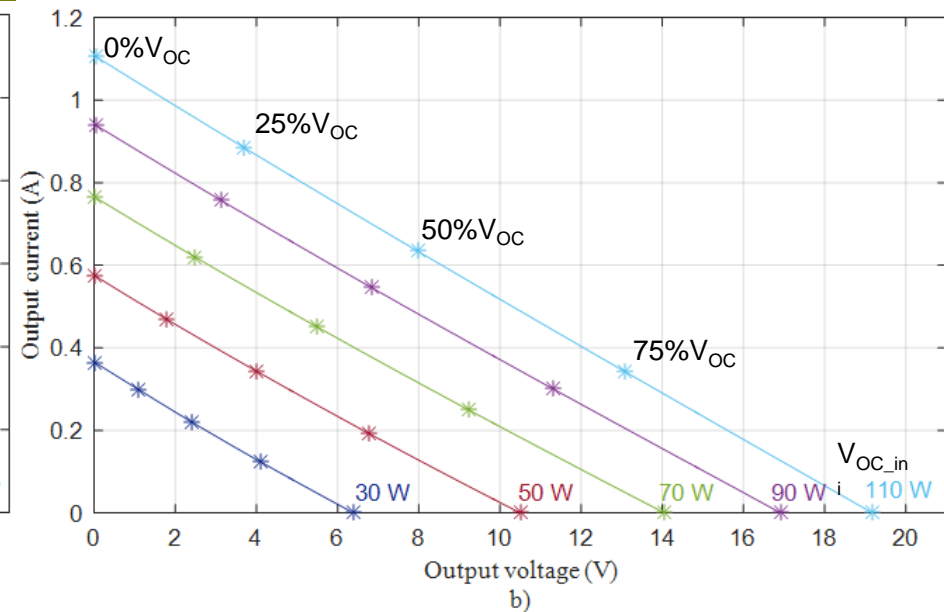
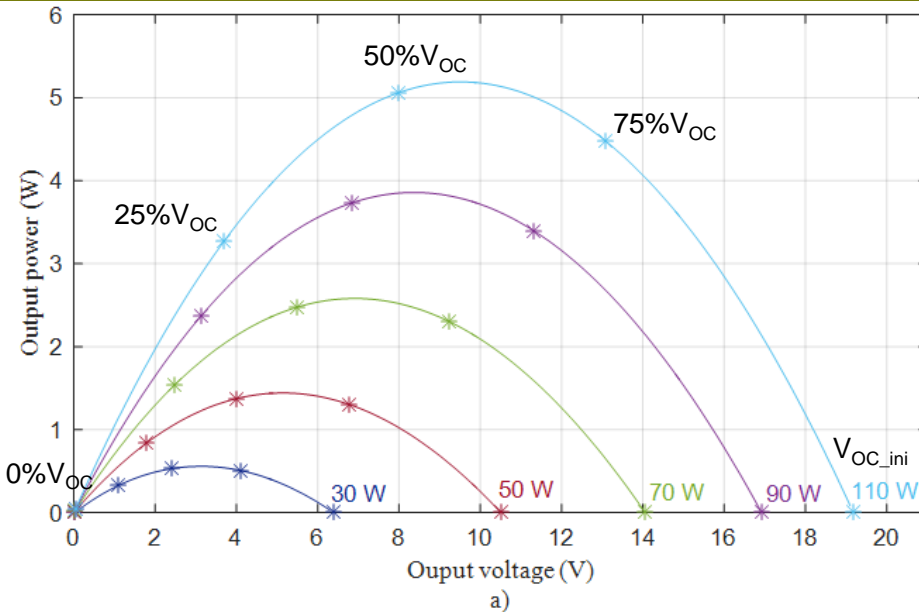
Constant heat characterization



Temperature difference variation in constant heat testing of thermoelectric module from open-circuit to short-circuit condition.

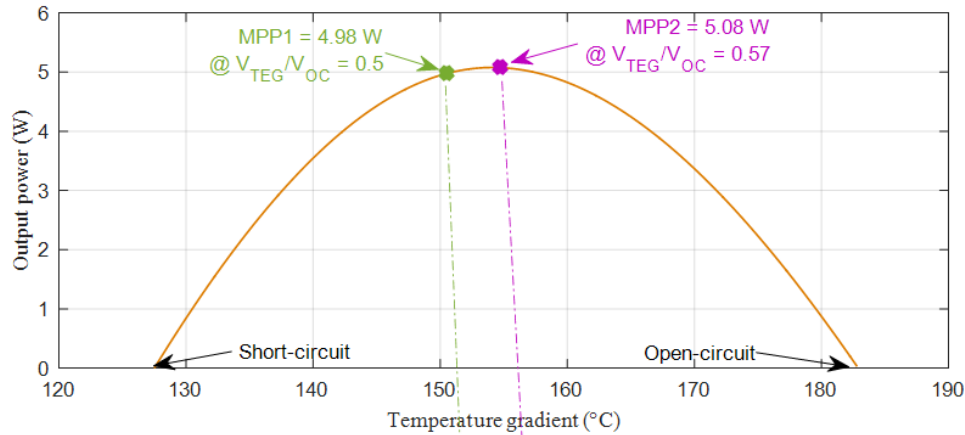


Constant heat characterization

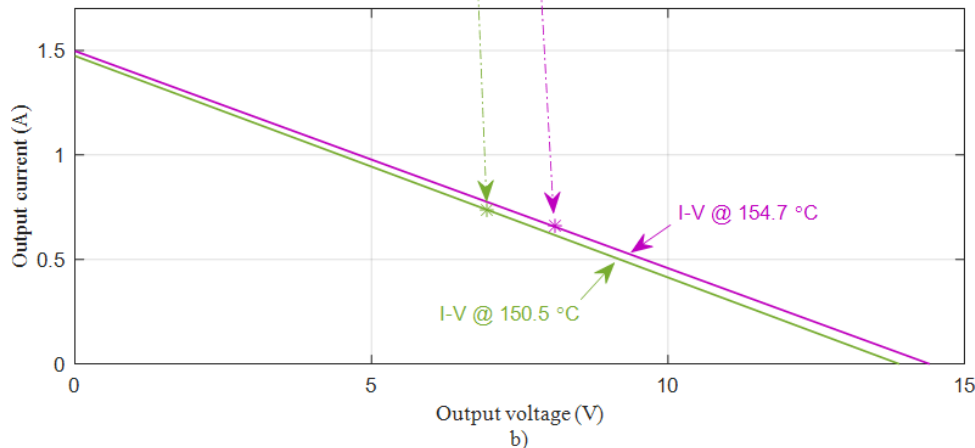


Output power versus output voltage and b) output current versus output voltage for GM250-241-10-16. The module was tested in constant heat starting at 30W to 110W with 20W increments.

- STEPS:**
1. Maintain constant heat flow through module (e.g. 110W) and reach thermal equilibrium.
 2. Measure V_{OC_ini} and set $V_{load} = \beta V_{OC_ini}$ where $\beta = \{75, 50, 25, 0\}$.
 3. Repeat V_{oc} measurements every few seconds and set V_{load} until thermal equilibrium, then measure and log voltage and current values.



Output power versus output voltage
(constant heat @ 130W)

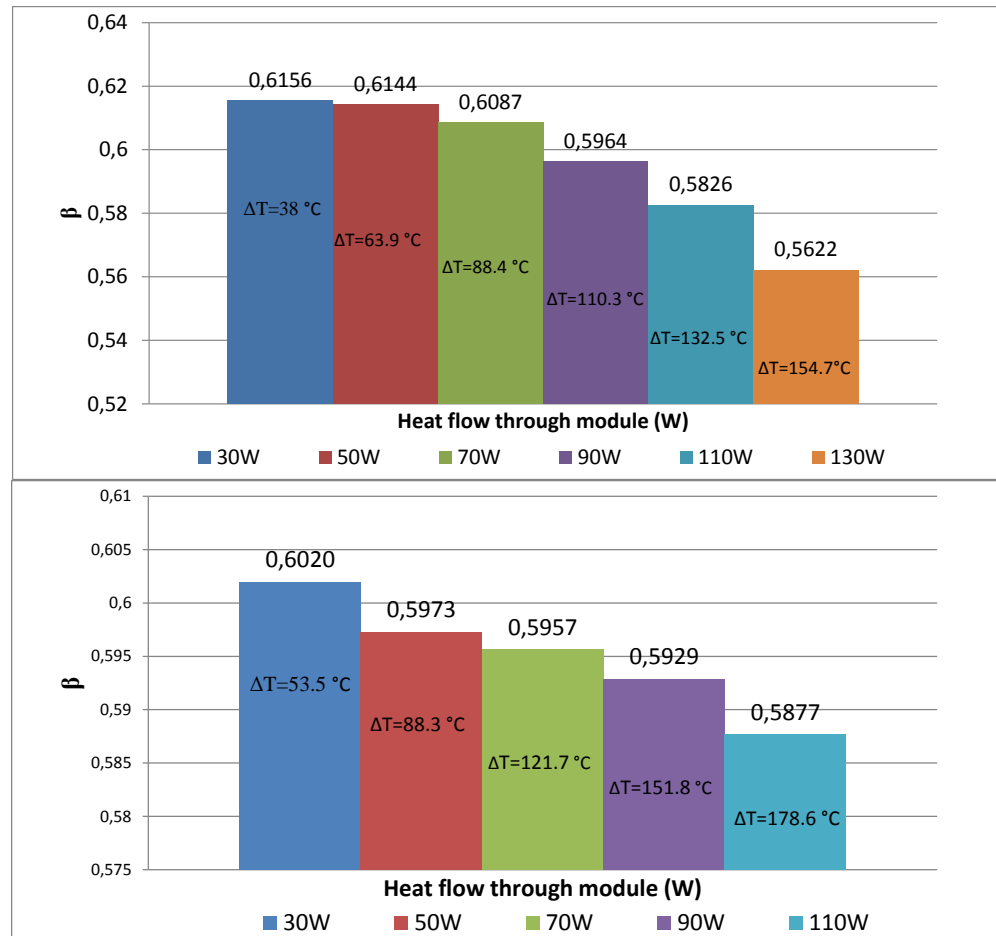


and b) output current versus output
voltage (constant temperature) for
GM250-241-10-12.



The fraction between the set voltage and V_{OC} (β) that sets the conditions to achieve MPP in constant heat

$$\beta = V_{load}/V_{oc}$$



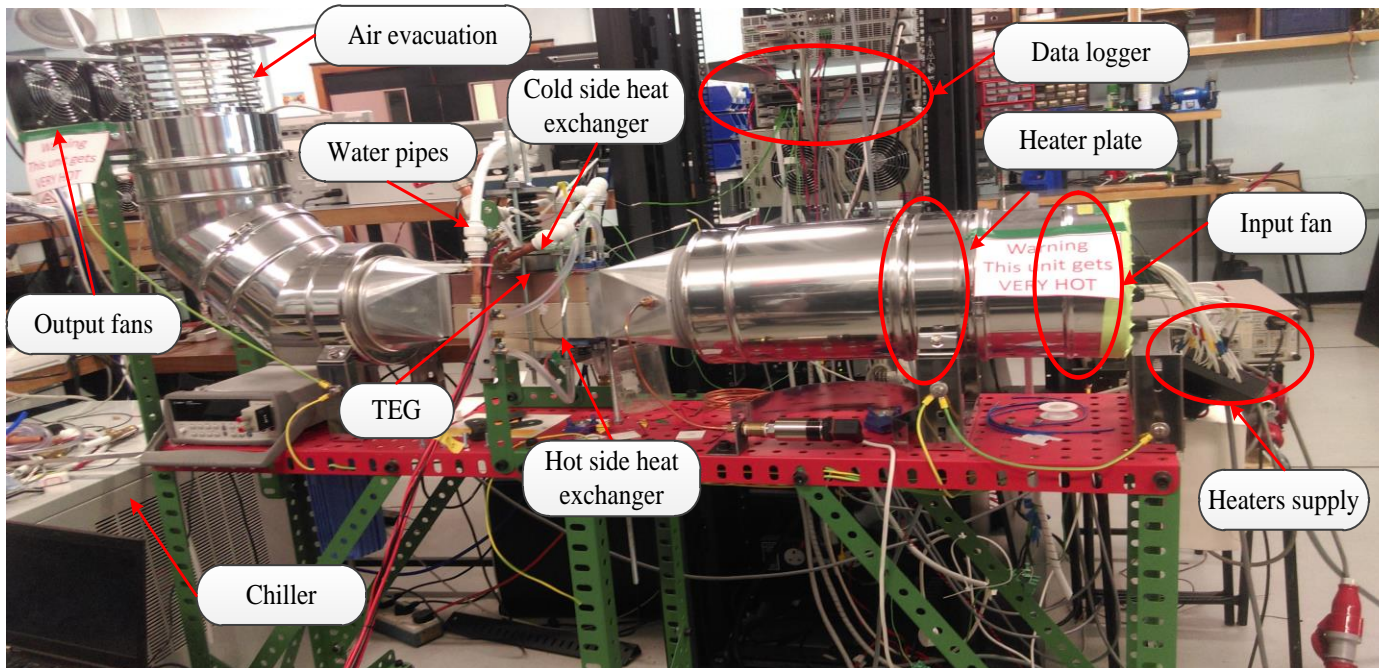
GM250-241-10-12
(40mmx40mmx1.2mm)
(short pellets)

ΔT on TEG faces

GM250-241-10-16
(40mmx40mmx1.6mm)
(longer pellets)



Partially controllable test system to validate real MPP concept

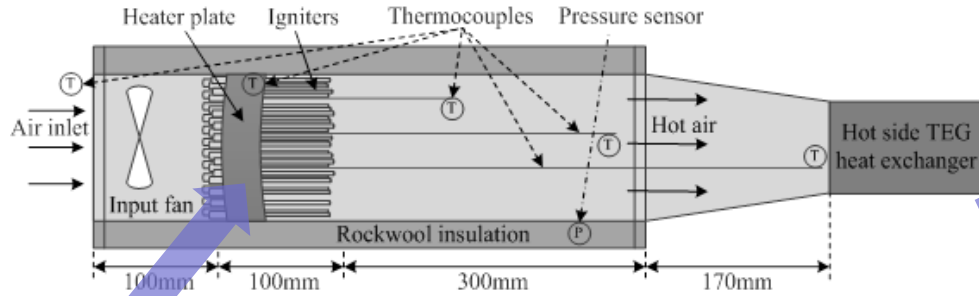




Hot gas system



Schematic of the heater side



Heater plate



Cold heat exchanger



TEG module assembly



Hot gas system - Results



Power increase: real MPP vs theoretical MPP (maximum power transfer theorem)

GM250-241-10-16				
Q_{teg} (W)	β	$P_{MPP,\beta=0.5}$ (W)	$P_{MPP,\beta\neq 0.5}$ (W)	PowerIncr. (%)
30	0.602	0.5307	0.5583	5.20
50	0.5973	1.3736	1.4421	4.98
70	0.5957	2.4766	2.5813	4.22
90	0.5929	3.7312	3.8584	3.41
110	0.5877	5.0577	5.1911	2.63

* $Q_{teg_max} = 145W$

GM250-241-10-12				
Q_{teg} (W)	β	$P_{MPP,\beta=0.5}$ (W)	$P_{MPP,\beta\neq 0.5}$ (W)	PowerIncr. (%)
30	0.6156	0.3699	0.3875	4.75
50	0.6144	0.9591	1.0101	5.31
70	0.6087	1.8138	1.9081	5.19
90	0.5964	2.7708	2.8827	4.03
110	0.5826	4.0509	4.1947	3.54
130	0.5622	4.9816	5.0769	1.91

* $Q_{teg_max} = 185W$

GM250-127-28-10				
Q_{teg} (W)	β	$P_{MPP,\beta=0.5}$ (W)	$P_{MPP,\beta\neq 0.5}$ (W)	PowerIncr. (%)
63	0.62	1.74	1.89	8.36
540	0.54	23.63	24.24	2.56

* $Q_{teg_max} = 560W$

This is the increase in the maximum power obtained from the TEG



Controllable TEG test system (one-module test fixture)

vs.

Uncontrollable TEG test system (multiple-module hot gas system)

- ❖ Temperature or heat control.
- ❖ Additional parameter measurements (*i.e.* pressure, hot side temperature)
- ❖ Automatic pressure control (fixed pressure at variable temperature).
- ❖ TEG characterization (usually better performance is obtained due to improved testing conditions).
- ❖ Hot heat exchangers dimensioned to fit module (thermal short-circuit avoided).
- ❖ Small size and weight.
- ❖ Slight influenced by changing atmospheric conditions (good test repeatability).
- ❖ Low total heat flux.

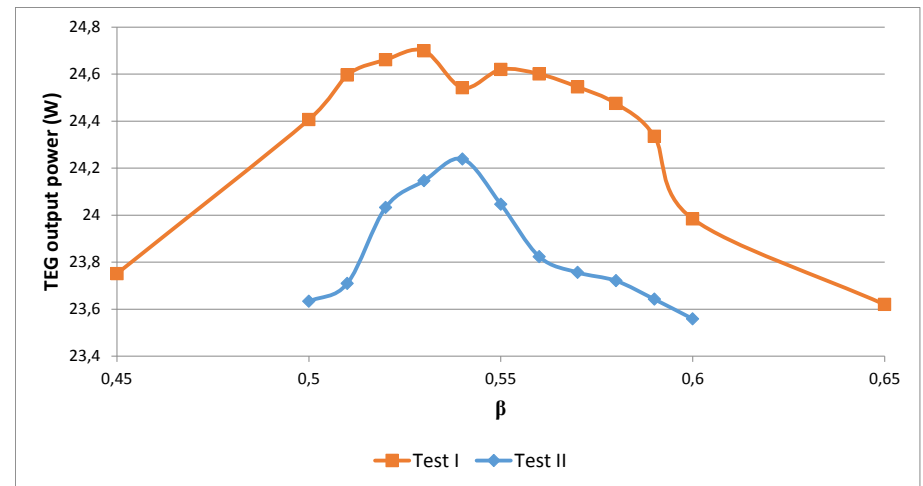
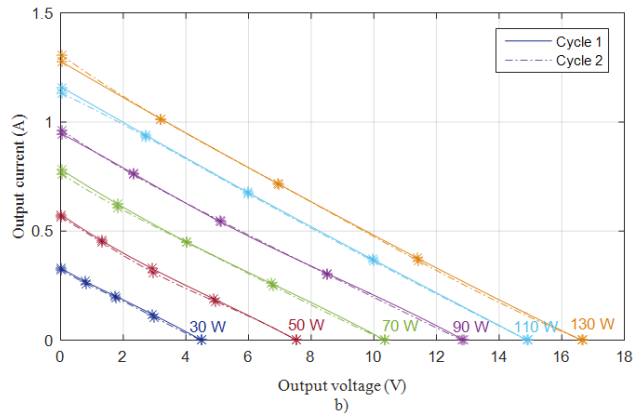
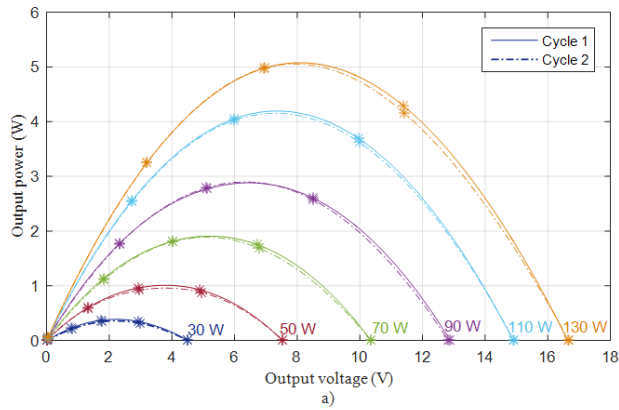
- ❖ Partial heat control.
- ❖ Fixed pressure upon assembly.
- ❖ Precise module characterization equivalent to its performance in TEG applications.
- ❖ Oversized hot heat exchanger (higher heat losses).
- ❖ Larger size and weight.
- ❖ Highly influenced by changing atmospheric conditions (more difficult to achieve test repeatability).
- ❖ High total heat flux.



Controllable TEG test system (one-module test fixture)

vs.

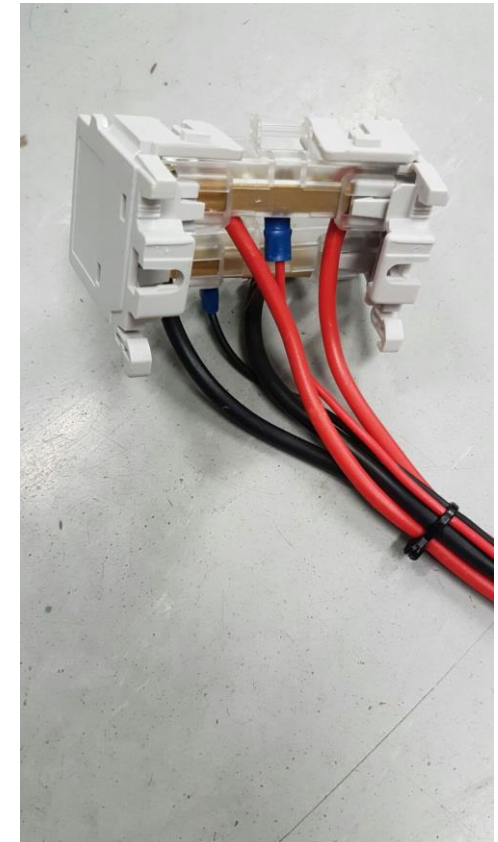
Uncontrollable TEG test system (multiple-module hot gas system)





TE module design

- ❖ Large pellets => high current => thick cables => wire losses (~2W losses @ 10A, 24W MPP – 4-wire sense, ~2m wire length).
- ❖ High current => high losses in converters, need of large heat sink for semiconductor devices => increased volume and weight.
- ❖ Low current => failure to start-up converters and difficulties in THP control.





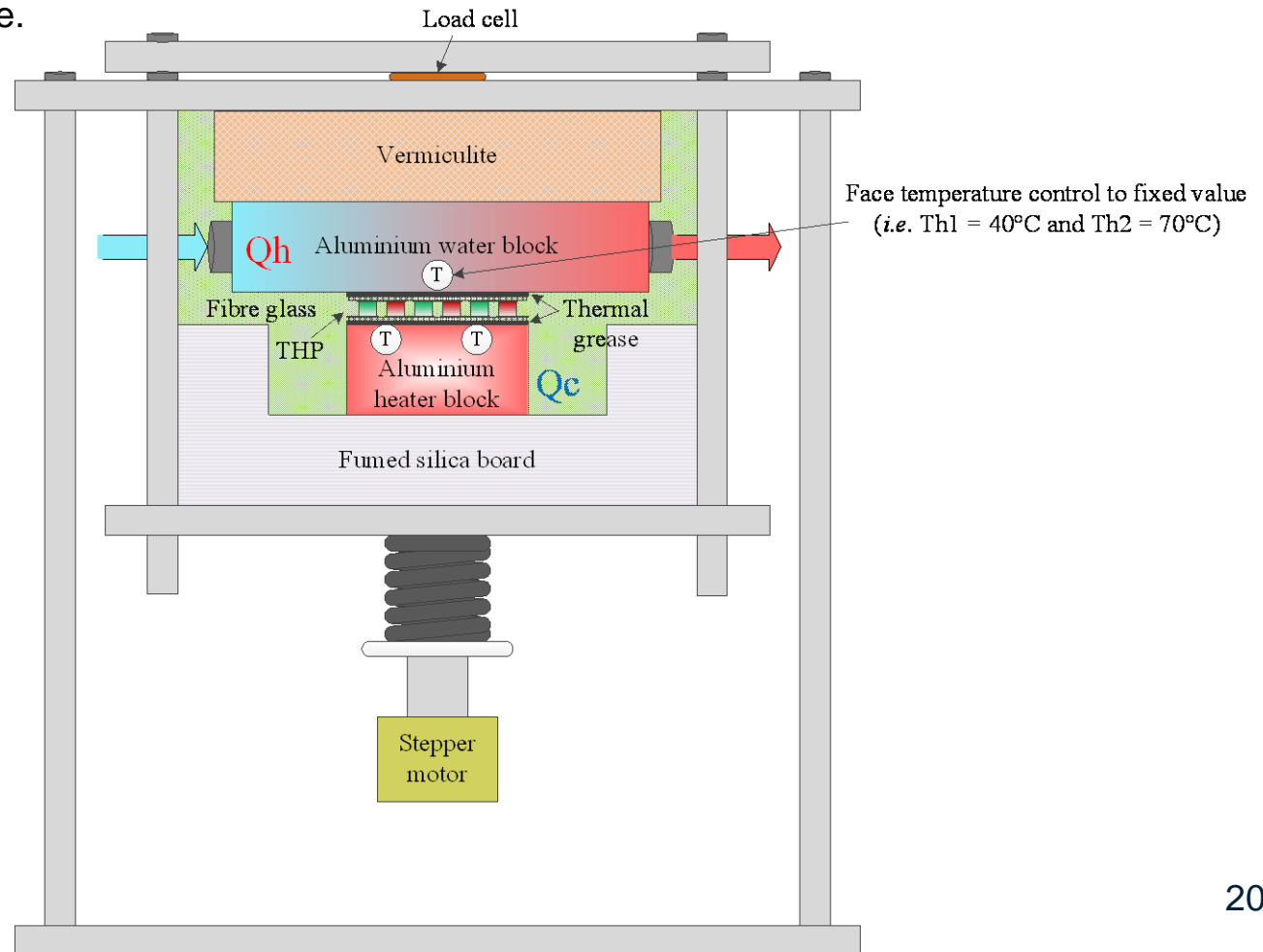
THERMOELECTRIC COOLING



THP characterization

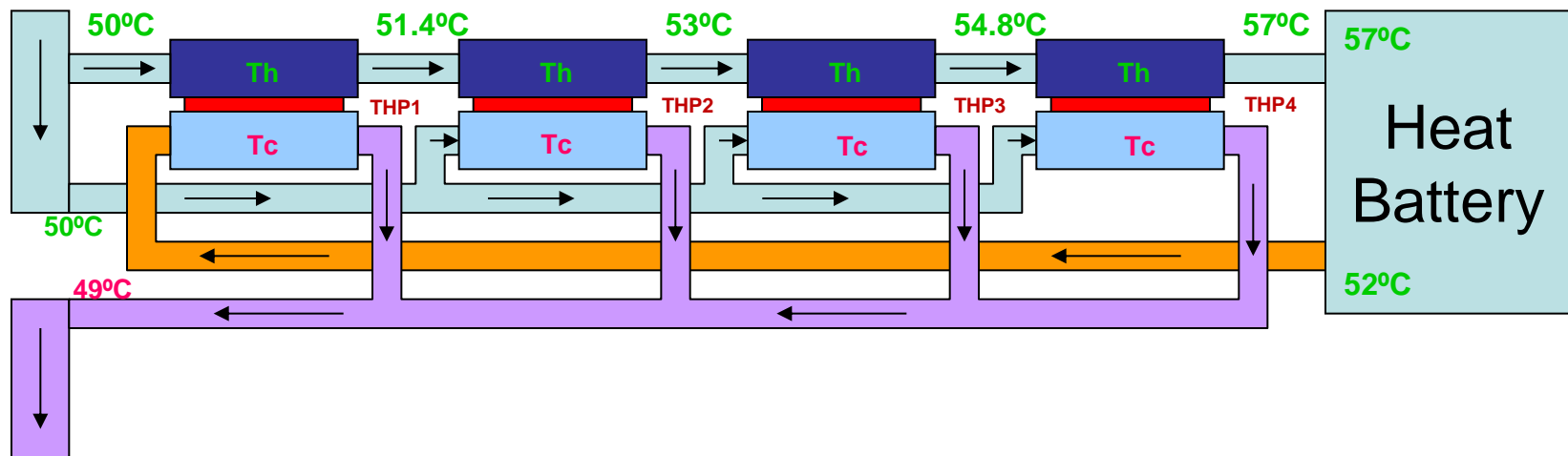


THP selected is a Thermonamic TEHP11256-0.3, 55x55mm module. Parameter characterization is performed using test fixture.





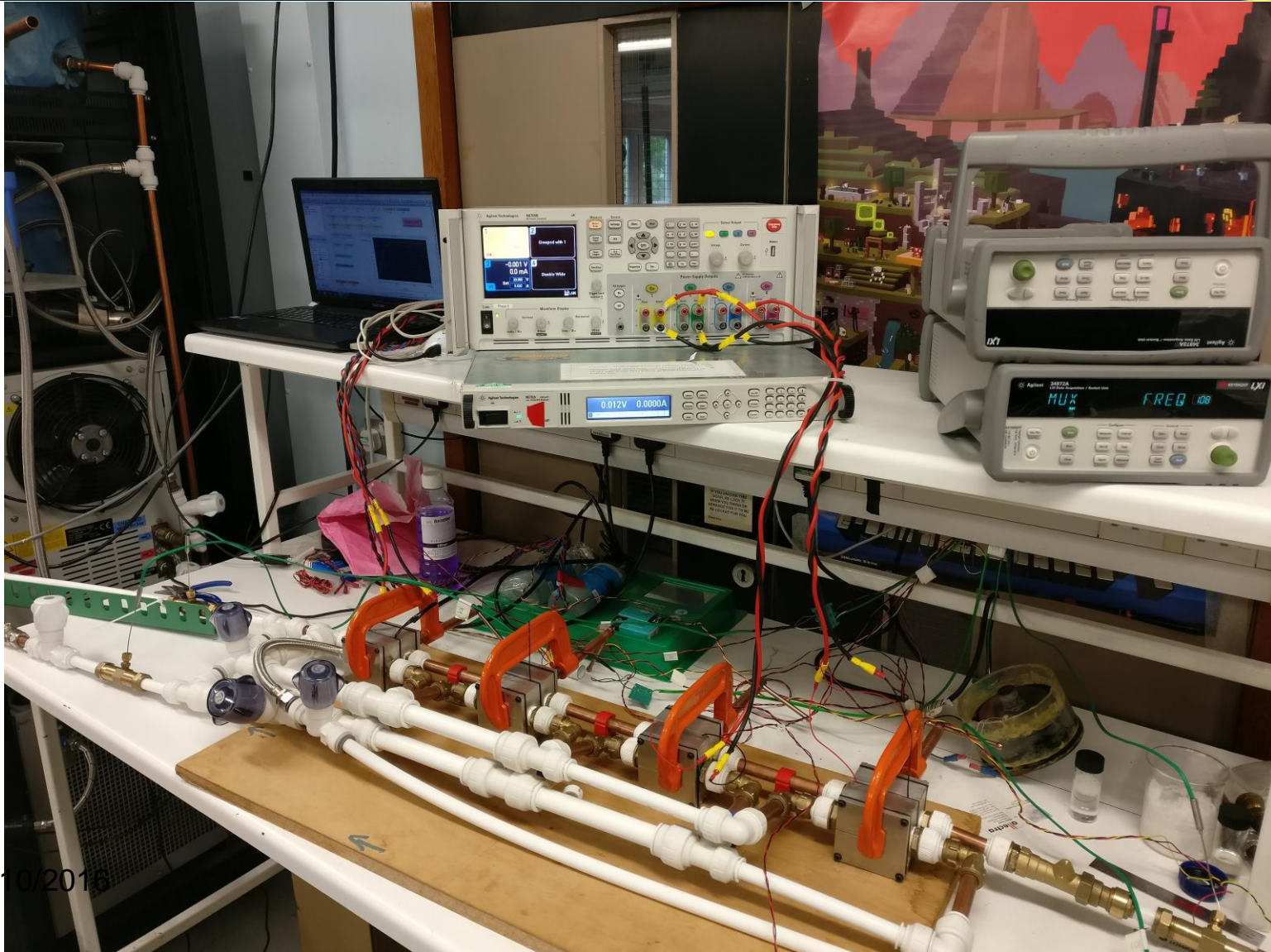
A series of cascaded heat pumps allows for water temperature to be controlled. The purpose is to boost the temperature and thermal energy to a phase change material based heat battery. Application: The input water of the heat amplifier system is previously used to cool PV panels.



This particular arrangement of pipework allows for the optimum ΔT and high COP across each stage.



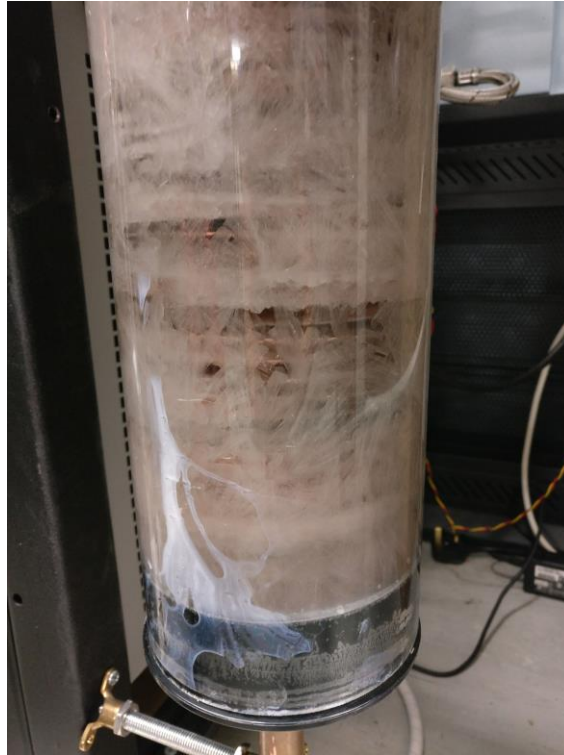
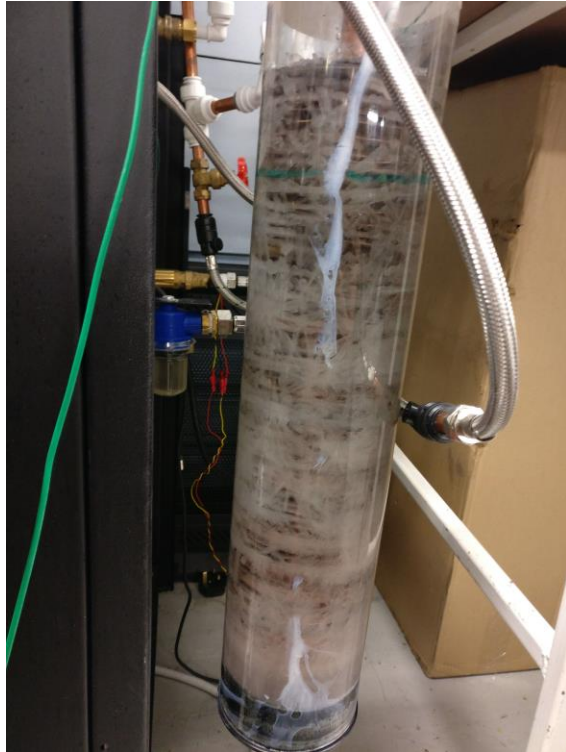
Heat amplifier system





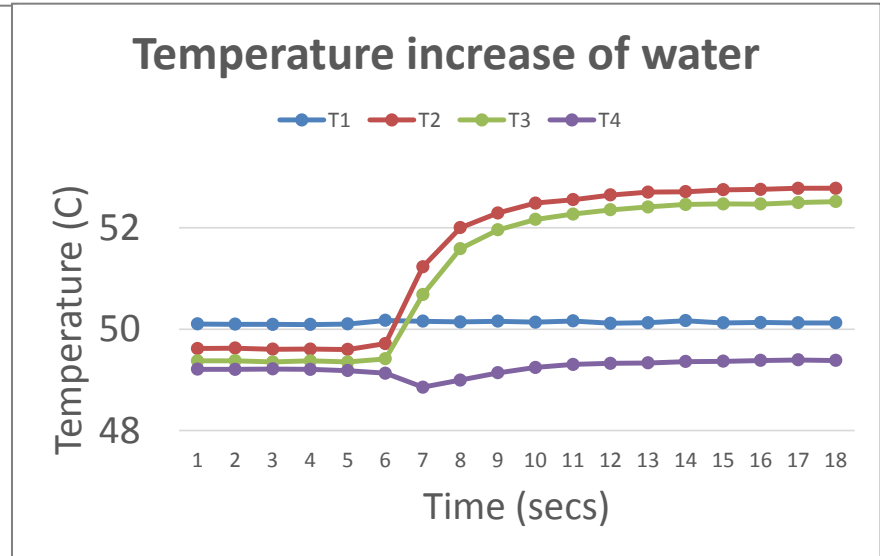
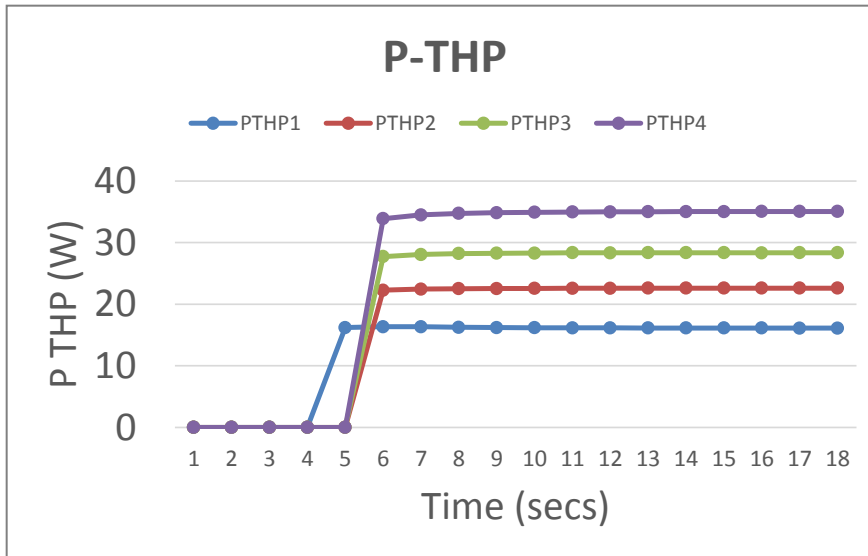
Heat battery

Calcium Chloride Hexahydrate





Heat amplifier system - Results



THP	1	2	3	4
COP	2.71	2.33	2.00	1.76



Heat amplifier system - Issues



- ❖ Parameters of the heat pumps for system operating at assumed fixed $T_h = 50^\circ\text{C}$ were extrapolated from characterizations at $T_h = 40^\circ\text{C}$ and $T_h = 70^\circ\text{C}$.
- ❖ Heat pump limited at 29A due to thickness of module wires (no datasheet).
- ❖ Initial system designed for 0.5 L/min.
- ❖ Flow meter specifications: 1 – 60 L/min.
- ❖ No flow control (chiller).
- ❖ High heat losses – difficult to insulate => experimentally measured temperature difference across the composite heat pump was half of that expected from calculations.



Thank you! Any Questions?

